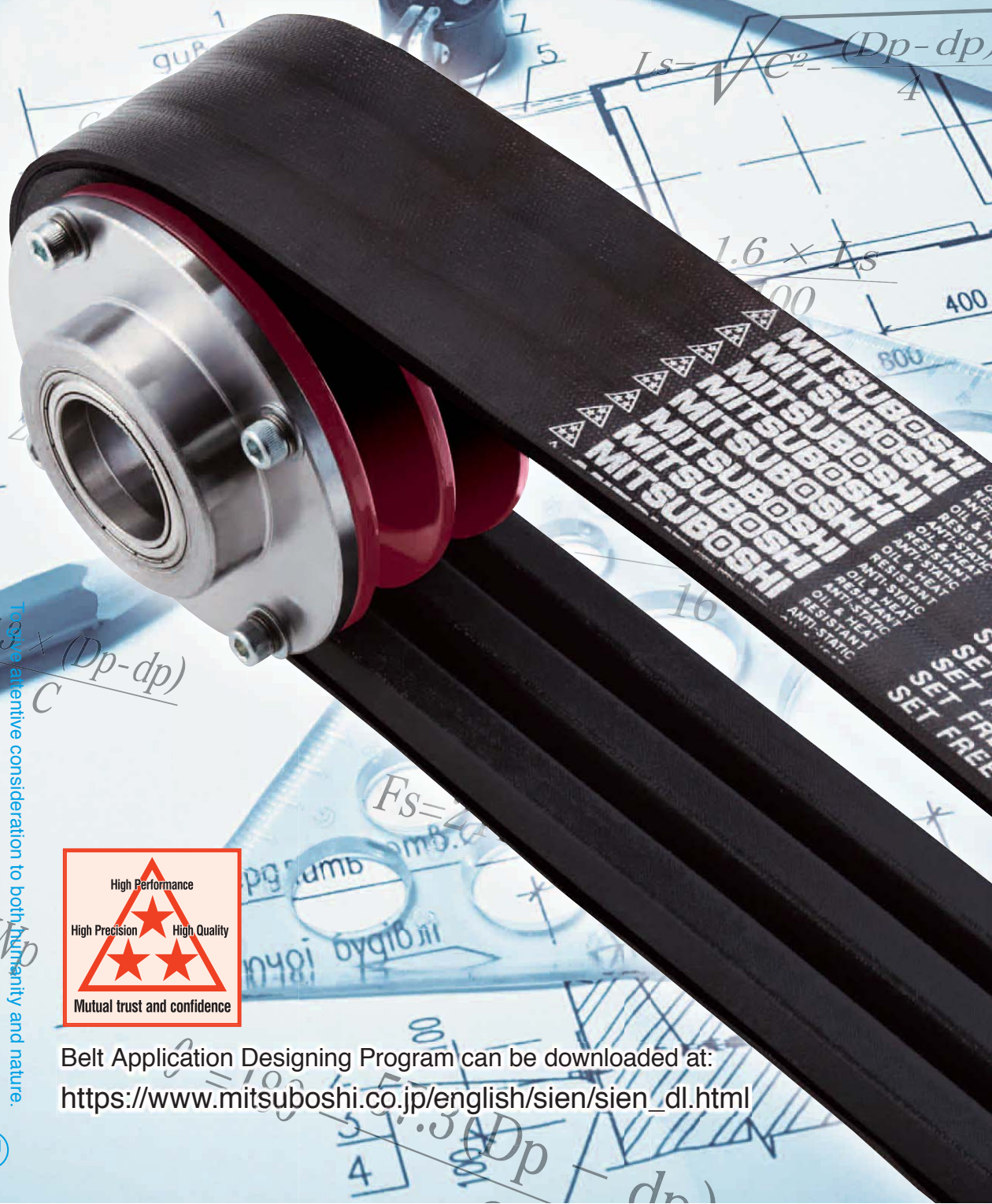


Design Manual V-Belt **DIN**

Classical V-Belts for DIN
Maxstar Wedge V-Belts for RMA / MPTA
Narrow V-Belts for DIN



MITSUBOSHI

®

Taking attentive consideration to both humanity and nature.



Belt Application Designing Program can be downloaded at:
https://www.mitsuboshi.co.jp/english/sien/sien_dl.html

Safety Precautions

Please read all the warnings!

- Please take all necessary precautions when using our products. Also, please review relevant product catalog and design documents, etc.

Significances of safety precautions are categorized as follows:

Signs	Meanings
 Danger	Imminently causing death or severe injury to the user who misuses products.
 Warning	Possibly causing death or severe injury to the user who misuses products.
 Caution	Possibly causing personal injury or property damage if misused.

Power Transmission Products

Use	
 Danger	● If you expect that a belt will fail and idle, free-run, or stop the system, thus causing a fatal or severe accident, please provide an extra safety device. ● Do not use a belt as a lifting or towing tool.
 Warning	● If you expect that static electricity will come from the power transmission belt system, thus causing fire or malfunction of the controller, use an antistatic belt and set a neutralization apparatus in the system.
 Caution	● Do not use a belt as an insulator. Contact us for information on insulation properties, which vary in belt type. ● For a belt that touches food directly, use one that complies with the applicable food hygiene law of your country. ● Do not modify a belt, or its quality and performance could deteriorate.

Function & Performance	
 Caution	● Do not use a belt beyond its capacity or for an application other than that specified by the catalog, design documents, etc. This can cause premature failure of the belt. ● If water, oil, chemical, paint, dust, etc. sticks to a belt or pulley, its power transmission could deteriorate and the belt may fail. ● A clogged belt makes louder noise during high-speed rotation. If this occurs, use a soundproof cover.

Storage & Transportation	
 Warning	● To store a heavy belt, use a suitable jig or stopper to prevent accidents such as belt toppling or tumbling.
 Caution	● Use suitable equipment to carry/handle a heavy belt or pulley. Otherwise, back injury may result. ● Do not put weight on or bend a belt forcibly to carry or store it. Otherwise, it will produce defects or scratches to the belt, resulting in damage. ● Store the belt in low humidity and a temperature range of -10°C to 40°C. Do not expose belts to direct sunlight.

Mounting & Operation	
 Danger	● Install a safety cover over rotating components including belt/ pulley. Otherwise, hair, gloves and clothing can become entangled in the belt/ pulley. If a belt/pulley breaks, fragments may cause injuries. ● Take the following precautions to maintain, inspect and replace a belt. 1)Turn off power and wait until the belt and pulley have stopped completely. 2)Secure machinery so that it may not move during belt removal. 3)Use caution : Do not unintentionally turn on power.
 Caution	● Use the same type of belts or pulleys per OEM specification. Use of a different type may cause premature failure. ● Misalignment of the pulleys can damage the belt and result in flange failure. Make proper adjustments to system. ● Loosen the belt tension when changing belts. Do not force or stretch a belt over the flange. Do not use a screw driver or other sharp objects into when replacing the belt as this will result in damage. ● Apply the appropriate belt tension as specified by the relevant catalog and design documents, etc. Inappropriate tension could result in damage of the belt and shaft. ● Take the following precautions to modify the pulley in use: 1)Remove burrs and maintain proper pulley angle; 2)Secure accurate dimensions after modification; 3)Maintain the pulley strength after modification. ● Before assembling the flange with the pulley, check for foreign materials between the pulley and flange. Fasten the flange with a caulking tool and so on. Inappropriate installation could result in the flange coming off.

Handling of Used items	
 Caution	● Do not burn belt, or hazardous gas could be produced.

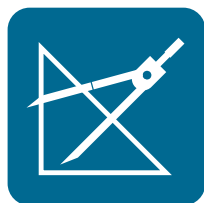


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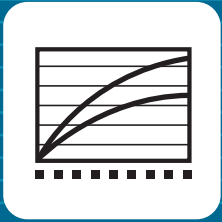


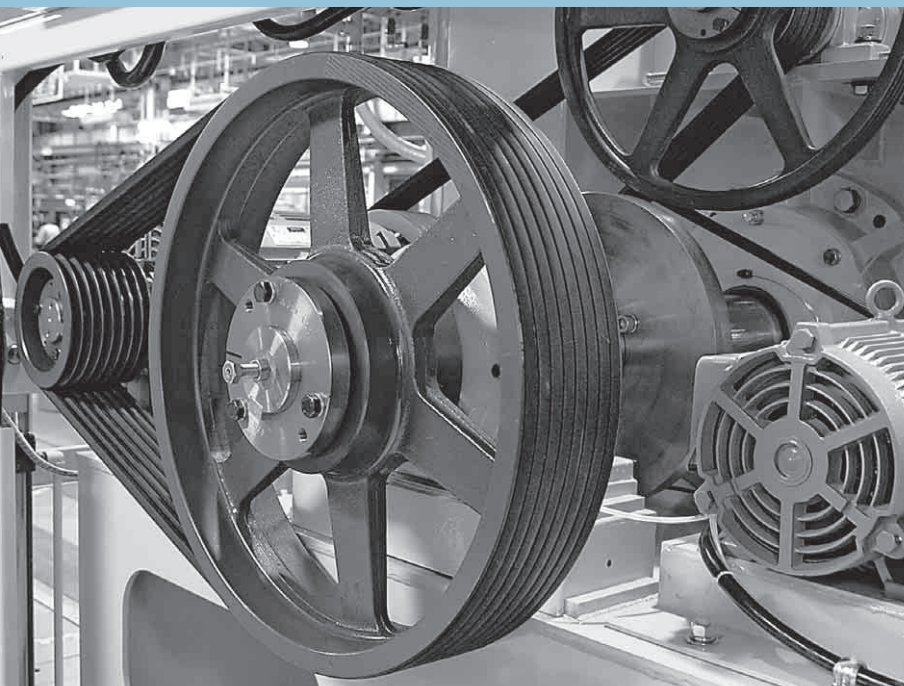
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1 Properties





1. Properties

Belt Construction

Product Classification

Standard Range

V-Belt pulleys

Belt construction

■ Wrapped V-Belts

"Wrapped" means that the V-Belt core is protected by cover fabric made of cotton or polyester.

The cover fabric is coated with rubber to reinforce the wear resistance.

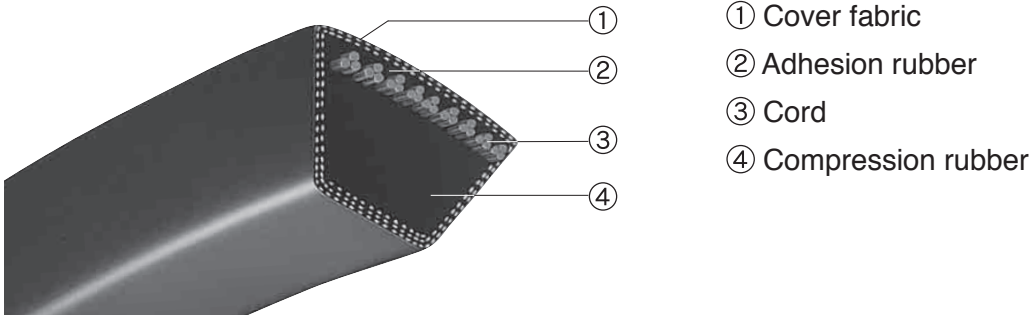


Fig. 1-1

1
Properties



■ Raw Edge V-Belts

Raw Edge V-Belts have no fabric on the belt sides.

The special rubber compound ensures greater wear resistance than Wrapped V-Belts.

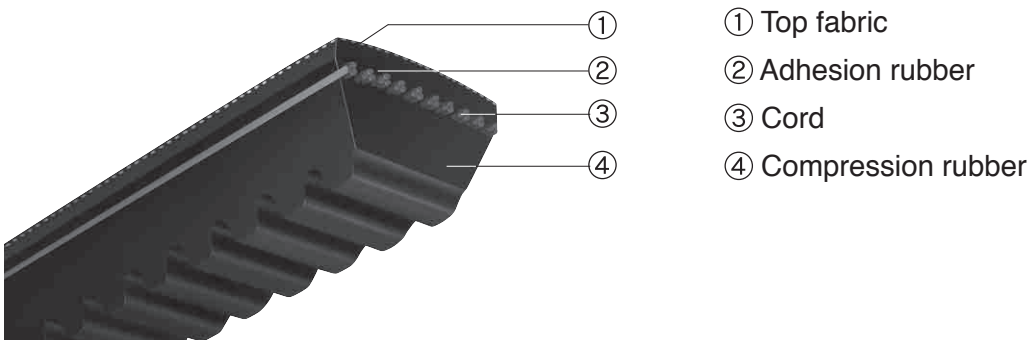


Fig. 1-2

Product Classification

■ Classical V-Belts for DIN 2215

Classical V-Belts are most widely used power transmission belts. Economical and easily obtained for replacement.



Fig. 1-3

Sections

Wrapped type : Z / A / B / C / D / E

Raw Edge type : ZX / AX / BX / CX

Working temperature

Wrapped type : -40 ~ +70°C

Raw Edge type : -30 ~ +90°C

Electrical conductivity

Suitable for ISO 1813

■ Maxstar Wedge V-Belts for RMA / MPTA

Maxstar Wedge V-Belts have double power transmission capacity of classical V-Belts due to greater wedge effect. It features high speed transmission of maximum 40 m/sec., energy saving, and compact design.

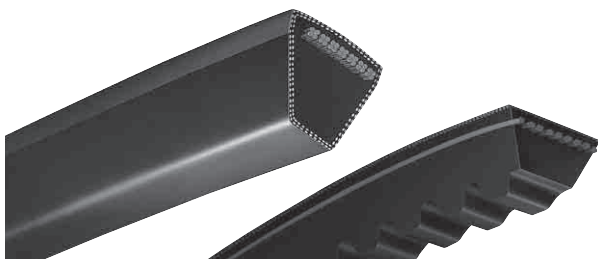


Fig. 1-4

Sections

Wrapped type : 3V / 5V / 8V

Raw Edge type : 3VX / 5VX

Working temperature

-30 ~ +90°C

Electrical conductivity

Suitable for RMA IP-3-3

■ Narrow V-Belts for DIN 7753 / ISO 4184

Narrow V-Belts enable space saving, high speed drive, and reduce the cost of operating and maintenance. It features oil / heat resistance and electrical conductivity.

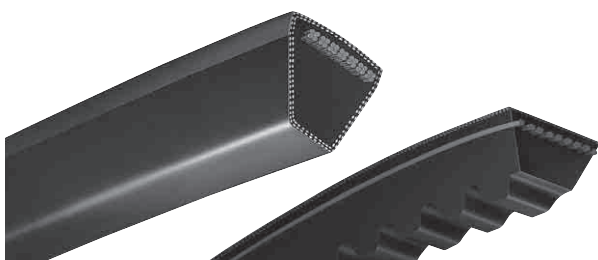


Fig. 1-5

Sections

Wrapped type : SPZ / SPA / SPB / SPC

Raw Edge type : SPZX / SPAX / SPBX / SPCX

Working temperature

-30 ~ +90°C

Electrical conductivity

Suitable for ISO 1813

1

Properties



Multi V-Belts

Multi V-Belts are made up of two or more standard V-Belts connected together at the top of the belts. Multi V-Belts have the advantage of preventing the belts from turning over or getting thrown off the drive even when belt vibration occurs.

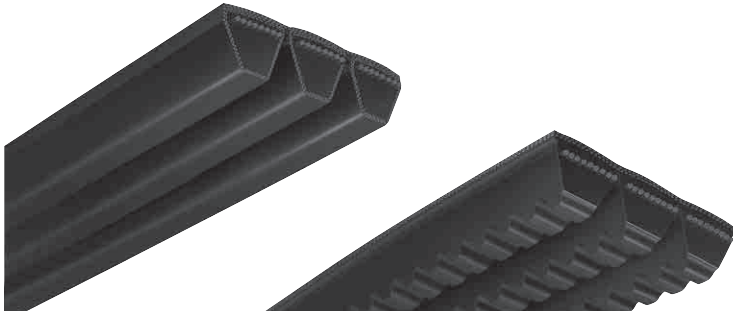


Fig. 1-6

Sections

Wrapped type : B / C / D
3V / 5V / 8V
SPB / SPC
Raw Edge type : BX / CX
3VX / 5VX

1
Properties



Double V-Belts

Double V-Belts are recommended for serpentine drives where the power must be transmitted by both the top and the bottom of the belts. Excellent flexibility in both directions.

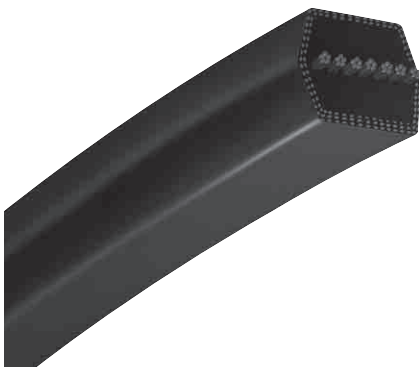


Fig. 1-7

Sections

AA / BB / CC

Perforated open-end V-Belts

Perforated open-end V-Belts are designed for easy installation with metal fasteners and a screwdriver. These belts are recommended for temporary use or when installation of the standard V-Belts is difficult.

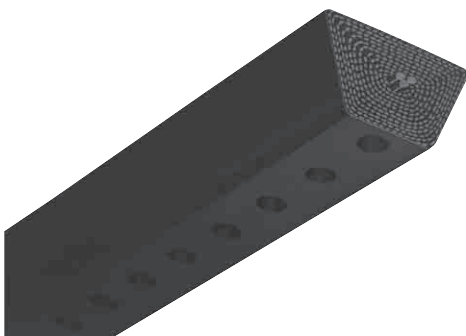


Fig. 1-8

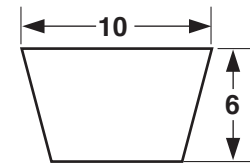
Sections

Z / A / B / C

Classical V-Belts for DIN 2215 / ISO 4184

Table 1-1

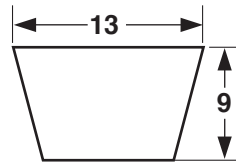
Z/10, ZX

**Belt indication****Z 50**

Cross Section

Belt Code(inch)

A/13, AX

**Belt indication****A 64**

Cross section

Belt Code(inch)

Z-Section**A-Section**

Belt Code	Inner length Li (mm)	Datum length Ld (mm)	Belt Code	Inner length Li (mm)	Datum length Ld (mm)	Belt Code	Inner length Li (mm)	Datum length Ld (mm)
20	515	537	20	508	538	71	1800	1830
21	530	552	21	535	565	72	1825	1855
22	560	582	22	560	590	73	1854	1884
23	585	607	23	585	615	74	1880	1910
24	610	632	24	610	640	75	1900	1930
25	630	652	25	630	660	76	1930	1960
26	660	682	26	660	690	77	1956	1986
27	685	707	27	686	716	78	1980	2010
28	710	732	28	710	740	79	2000	2030
29	730	752	29	730	760	80	2032	2062
30	765	787	30	767	797	81	2060	2090
31	790	812	31	787	817	82	2083	2113
32	820	842	32	813	843	83	2110	2140
33	840	862	33	838	868	84	2134	2164
34	865	887	34	864	894	85	2160	2190
35	890	912	35	889	919	86	2180	2210
36	915	937	36	914	944	87	2210	2240
37	940	962	37	940	970	88	2240	2270
38	965	987	38	965	995	89	2260	2290
39	990	1012	39	990	1020	90	2286	2316
40	1016	1038	40	1016	1046	91	2310	2340
41	1041	1063	41	1041	1071	92	2337	2367
42	1060	1082	42	1060	1090	93	2360	2390
43	1090	1112	43	1100	1130	94	2388	2418
44	1120	1142	44	1120	1150	95	2413	2443
45	1140	1162	45	1143	1173	96	2438	2468
46	1165	1187	46	1168	1198	97	2464	2494
47	1194	1216	47	1200	1230	98	2500	2530
48	1225	1247	48	1220	1250	99	2515	2545
49	1250	1272	49	1250	1280	100	2540	2570
50	1270	1292	50	1270	1300	102	2591	2621
			51	1300	1330	105	2667	2697
			52	1320	1350	108	2743	2773
			53	1350	1380	110	2800	2830
			54	1375	1405	112	2845	2875
			55	1400	1430	115	2921	2951
			56	1422	1452	118	3000	3030
			57	1450	1480	120	3048	3078
			58	1475	1505	122	3100	3130
			59	1500	1530	125	3175	3205
			60	1525	1555	128	3250	3280
			61	1550	1580	130	3300	3330
			62	1575	1605	135	3425	3455
			63	1600	1630	140	3550	3580
			64	1625	1655	150	3810	3840
			65	1650	1680	160	4064	4094
			66	1676	1706	170	4318	4348
			67	1700	1730	180	4572	4602
			68	1725	1755			
			69	1750	1780			
			70	1775	1805			

Size range: 20" ~ 93"

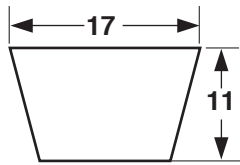
Size range: 20" ~ 360"



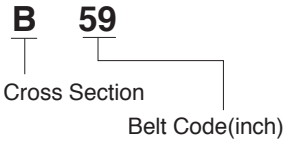
Classical V-Belts for DIN 2215 / ISO 4184

Table 1-2

B/17, BX



Belt indication



B-Section

Belt Code	Inner length Li (mm)	Datum length Ld (mm)	Belt Code	Inner length Li (mm)	Datum length Ld (mm)
25	630	673	75	1900	1943
26	660	703	76	1930	1973
27	686	729	77	1950	1993
28	710	753	78	1981	2024
29	735	778	79	2000	2043
30	762	805	80	2032	2075
31	785	828	81	2060	2103
32	813	856	82	2083	2126
33	838	881	83	2100	2143
34	865	908	84	2134	2177
35	889	932	85	2160	2203
36	915	958	86	2200	2243
37	940	983	87	2210	2253
38	965	1008	88	2240	2283
39	991	1034	89	2260	2303
40	1016	1059	90	2286	2329
41	1040	1083	91	2300	2343
42	1060	1103	92	2337	2380
43	1090	1133	93	2360	2403
44	1120	1163	94	2388	2431
45	1150	1193	95	2413	2456
46	1175	1218	96	2438	2481
47	1200	1243	97	2465	2508
48	1215	1258	98	2500	2543
49	1250	1293	99	2515	2558
50	1275	1318	100	2540	2583
51	1300	1343	102	2600	2643
52	1320	1363	105	2667	2710
53	1350	1393	106	2700	2743
54	1372	1415	108	2750	2793
55	1400	1443	110	2800	2843
56	1425	1468	112	2845	2888
57	1450	1493	115	2921	2964
58	1475	1518	118	3000	3043
59	1500	1543	120	3048	3091
60	1525	1568	122	3099	3142
61	1550	1593	125	3175	3218
62	1575	1618	128	3250	3293
63	1600	1643	130	3300	3343
64	1625	1668	132	3350	3393
65	1650	1693	135	3425	3468
66	1675	1718	138	3500	3543
67	1700	1743	140	3550	3593
68	1725	1768	145	3675	3718
69	1750	1793	150	3810	3853
70	1775	1818	155	3937	3980
71	1800	1843	160	4065	4108
72	1825	1868	Size range: 23" ~ 660"		
73	1850	1893			
74	1880	1923			

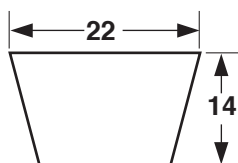
Available size for Raw Edge Cogged V-Belts BX



Classical V-Belts for DIN 2215 / ISO 4184

Table 1-3

C/22, CX



Belt indication

C 93

Cross Section Belt Code(inch)

C-Section

Belt Code	Inner length Li (mm)	Datum length Ld (mm)	Belt Code	Inner length Li (mm)	Datum length Ld (mm)
40	1000	1052	96	2438	2490
42	1075	1127	97	2465	2517
45	1150	1202	98	2500	2552
48	1220	1272	99	2525	2577
50	1270	1322	100	2540	2592
51	1295	1347	102	2591	2643
52	1320	1372	105	2667	2719
53	1350	1402	108	2750	2802
54	1375	1427	110	2800	2852
55	1400	1452	112	2845	2897
56	1425	1477	115	2921	2973
57	1450	1502	118	3000	3052
58	1475	1527	120	3050	3102
59	1500	1552	122	3100	3152
60	1525	1577	125	3175	3227
61	1550	1602	128	3250	3302
62	1575	1627	130	3300	3352
63	1600	1652	132	3350	3402
64	1625	1677	138	3500	3552
65	1650	1702	140	3550	3602
66	1675	1727	142	3600	3652
67	1700	1752	148	3750	3802
68	1725	1777	150	3810	3862
69	1750	1802	155	3937	3989
70	1775	1827	160	4064	4116
71	1800	1852	170	4318	4370
72	1829	1881	180	4572	4624
73	1854	1906	190	4825	4877
74	1880	1932	195	4950	5002
75	1900	1952	197	5000	5052
76	1930	1982	200	5080	5132
77	1956	2008	210	5334	5386
78	1981	2033	220	5600	5652
79	2000	2052	230	5842	5894
80	2032	2084	240	6096	6148
81	2060	2112	250	6350	6402
82	2083	2135	260	6600	6652
83	2108	2160	270	6858	6910
84	2135	2187	Size range: 37" ~ 660"		
85	2159	2211			
86	2184	2236			
87	2210	2262			
88	2240	2292			
89	2261	2313			
90	2286	2338			
91	2311	2363			
92	2337	2389			
93	2360	2412			
94	2388	2440			
95	2413	2465			

: Available size for Raw Edge Cogged V-Belts CX

1

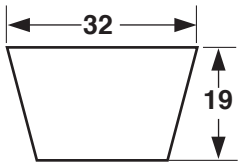
Properties



Classical V-Belts for DIN 2215 / ISO 4184

Table 1-4

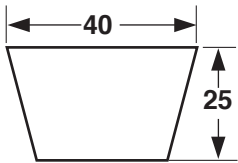
D/32



Belt indication
D 120

Cross Section
Belt Code(inch)

E/40



Belt indication
E 180

Cross Section
Belt Code(inch)

D-Section

Belt Code	Inner length Li (mm)	Datum length Ld (mm)
100	2540	2615
105	2675	2750
110	2800	2875
115	2925	3000
120	3048	3123
130	3300	3375
135	3425	3500
140	3550	3625
150	3810	3885
155	3925	4000
160	4060	4135
165	4200	4275
170	4320	4395
180	4570	4645
190	4825	4900
200	5080	5155
210	5330	5405
220	5600	5675
230	5850	5925
240	6096	6171
250	6350	6425
260	6600	6675
270	6850	6925
280	7100	7175
300	7620	7695
310	7875	7950
330	8380	8455
360	9144	9219

Size range: 100" ~ 660"

E-Section

Belt Code	Inner length Li (mm)	Datum length Ld (mm)
180	4575	4657
210	5300	5382
240	6100	6182
270	6850	6932
300	7620	7702
330	8380	8462
360	9150	9232
390	9900	9982
420	10670	10752

Size range: 144" ~ 660"



Cross section Dimension of Classical V-Belts for DIN

Table 1-5

Section	ISO 4184 DIN 2215	Z 10	A 13	B 17	C 22	D 32	E 40
Top belt width	b_o (mm)	10	13	17	22	32	40
Datum width	b_d (mm)	8.5	11	14	19	27	32
Height of belt	h (mm)	6	9	11	14	19	25
Datum length	$L_d \div L_i+$ (mm)	22	30	43	52	75	82
Distance down to datum line	h_d (mm)	2.5	3.3	4.2	5.7	8.1	12
Outer length	$L_a \div L_i+$ (mm)	38	50	69	88	126	157
Recommended minimum Pulley datum diameter	dd (mm)	45 (40)	71 (63)	112 (90)	180 (140)	315	450
Recommended maximum Belt speed	V (m/s)	30					

() : Figure of recommended minimum Pulley datum diameter for Raw Edge Cogged type.

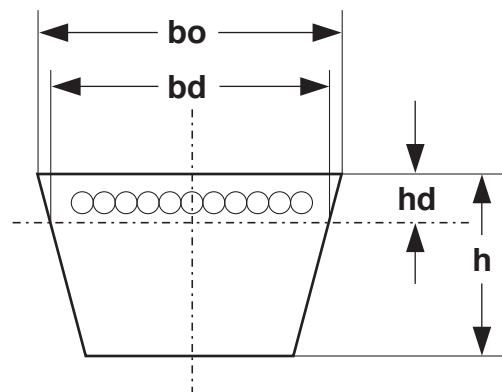
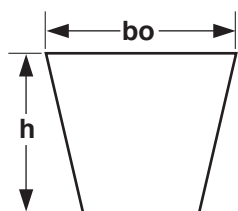


Fig. 1-9



Maxstar Wedge V-Belts for RMA / MPTA



Section	bo (mm)	h (mm)
3V/9N	9.5	8.0
5V/15N	16.0	13.5
8V/25N	25.5	23.0

Belt type	3VX		5VX	
Belt length range(inch)	25~95	96~270	25~95	96~270
bo (mm)	9.9	8.6	16.6	15.7
h (mm)	8.7	9.0	13.5	13.7

Belt indication

5V 1000

Cross Section

Belt Code(inch×10)

Table 1-6

3V(3VX)-Section		5V(5VX)-Section		8V-Section	
Belt Code	Effective length Le (mm)	Belt Code	Effective length Le (mm)	Belt Code	Effective length Le (mm)
250	635	500	1270	1000	2540
265	673	530	1346	1060	2692
280	711	560	1422	1120	2845
300	762	600	1524	1180	2997
315	800	630	1600	1250	3175
335	851	670	1702	1320	3353
355	902	710	1803	1400	3556
375	953	750	1905	1500	3810
400	1016	800	2032	1600	4064
425	1080	850	2159	1700	4318
450	1143	900	2286	1800	4572
475	1207	950	2413	1900	4826
500	1270	1000	2540	2000	5080
530	1346	1060	2692	2120	5385
560	1422	1120	2845	2240	5690
600	1524	1180	2997	2360	5994
630	1600	1250	3175	2630	5994
670	1702	1320	3353	2500	6350
710	1803	1400	3556	2650	6731
750	1905	1500	3810	2800	7112
800	2032	1600	4064	3000	7620
850	2159	1700	4318	3150	8001
900	2286	1800	4572	3350	8509
950	2413	1900	4826	3550	9017
1000	2540	2000	5080	3750	9525
1060	2692	2120	5385	4000	10160
1120	2845	2240	5690	4250	10795
1180	2997	2360	5994	4500	11430
1250	3175	2500	6350	4750	12065
1320	3353	2650	6731	5000	12700
1400	3556	2800	7112	5600	14224
		3000	7620	6000	15240
		3150	8001		
		3350	8509		
		3550	9017		

 : Available size for Raw Edge Cogged V-Belts 3VX & 5VX



Cross section dimension of Narrow V-Belts for DIN & Maxstar Wedge V-Belts for RMA / MPTA

Table 1-8

Section		SPZ	SPA	SPB	SPC	3V	5V	8V
Top belt width	bo (mm)	9.7	12.7	16.3	22.0	9.5	16.0	25.5
Datum width	bd (mm)	8.5	11.0	14.0	19.0			
Height of belt	h (mm)	8.0	10.0	13.0	18.0	8.0	13.5	23.0
Inner length	Li ÷ Ld- (mm)	38	45	60	83			
Outer length	La ÷ Ld+ (mm)	13	18	22	30			
Distance down to datum line	hd (mm)	2.0	2.8	3.5	4.8			
Recommended minimum Pulley datum diameter	dd (mm)	63 (56)	90 (71)	140 (112)	224 (180)	63 (56)	140 (112)	335
Recommended maximum Belt speed	V (m/s)	40						

() : Recommended minimum Pulley datum diameter for Raw Edge Cogged type.

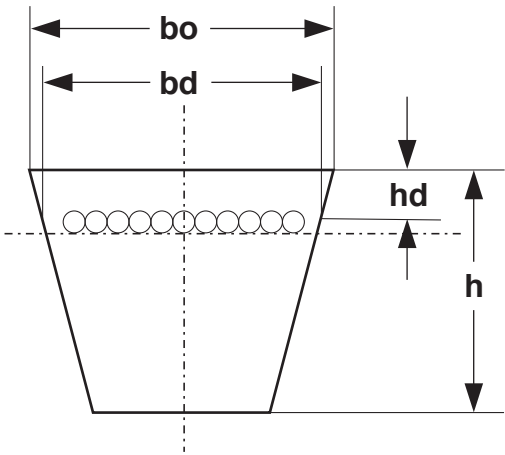


Fig. 1-10

1

Properties

Double V-Belts for DIN 7722/ISO 5289/RMA IP-21

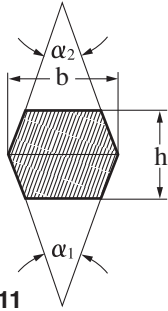


Fig. 1-11

Dimension of Double V-Belts

Table 1-9

Section	AA	BB	CC
Belt width b (mm)	12.5	15.5	22.0
Belt height h (mm)	10.0	13.0	17.0
Belt angle α_1 (°)	40	40	40
Belt angle α_2 (°)	40	40	40
Recommended minimum pulley diameter (mm)	75	130	230

AA Cross Section Table 1-10

Belt Code	Pitch length	
	inch	mm
AA 41	43.1	1095
AA 46	48.1	1222
AA 49	51.1	1298
AA 51	53.1	1349
AA 52	54.1	1374
AA 53	55.1	1400
AA 55	57.1	1450
AA 60	62.1	1577
AA 72	74.1	1882
AA 300	301.1	7648

BB Cross Section Table 1-11

Belt Code	Pitch length	
	inch	mm
BB 54	56.9	1445
BB 55	57.9	1471
BB 59	61.9	1572
BB 60	62.9	1598
BB 71	73.9	1877
BB 400	401.4	10196

CC Cross Section Table 1-12

Belt Code	Pitch length	
	inch	mm
CC 71	75.2	1910
CC 500	502.2	12756

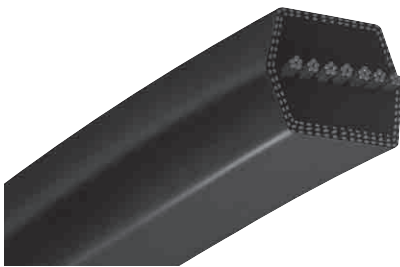


Fig. 1-12



Perforated Open-End V-Belts

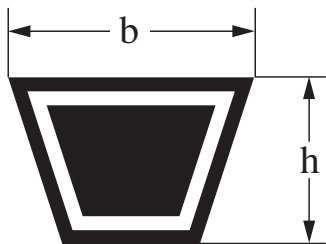


Fig. 1-13

Dimension of Perforated Open-End V-Belts				Table 1-13
Section	Z	A	B	C
Belt width b (mm)	10.0	12.5	16.5	22.0
Belt height h (mm)	5.5	9.0	11.0	14.0
Belt angle $(^\circ)$	40	40	40	40
Length per roll (m)	100	100	100	100
Hole diameter (mm)	2.0	2.5	3.0	4.0
Hole pitch (mm)	5.0	8.0	9.0	11.0

1

Properties

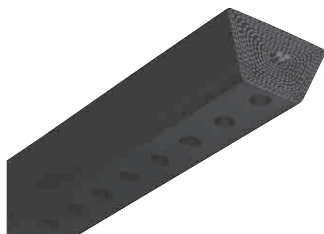


Fig. 1-14

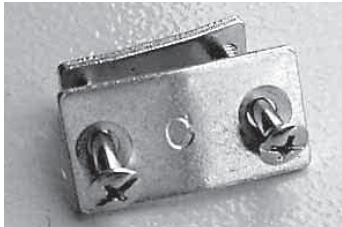


Fig. 1-15 (Metal fastener)

1

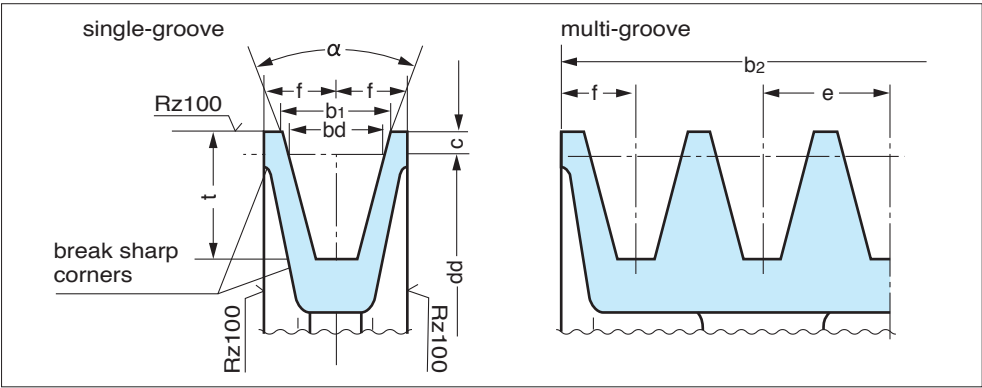
Properties



Pulley for Classical V-Belts & Narrow V-Belts for DIN

Pulley for Classical V-belts & Narrow V-Belts complies with DIN 2217 Part 1.

Fig. 1-16



1
Properties

Dimensions (mm)

Table 1-14

Belt section	Datum diameter : dd	α (°)	b1	bd	c	t	e	f
Z, ZX SPZ, SPZX	$dd \leq 80$ $80 < dd$	34 38	9.7	8.5	2.0	$11^{+0.6}_{-0}$	12.0 ± 0.3	8.0 ± 0.6
A, AX SPA, SPAX	$dd \leq 118$ $118 < dd$	34 38	12.7	11.0	2.8	$14^{+0.6}_{-0}$	15.0 ± 0.3	10.0 ± 0.6
B, BX SPB, SPBX	$dd \leq 190$ $190 < dd$	34 38	16.3	14.0	3.5	$18^{+0.6}_{-0}$	19.0 ± 0.4	12.5 ± 0.8
C, CX SPC, SPCX	$dd \leq 315$ $315 < dd$	34 38	22.0	19.0	4.8	$24^{+0.6}_{-0}$	25.5 ± 0.5	17.0 ± 1.0
D	$dd \leq 500$ $500 < dd$	36 38	32.0	27.0	8.1	$28^{+0.6}_{-0}$	37.0 ± 0.6	24.0 ± 2.0
E	$dd \leq 630$ $630 < dd$	36 38	40.0	32.0	12.0	$33^{+0.6}_{-0}$	44.5 ± 0.7	29.0 ± 2.0

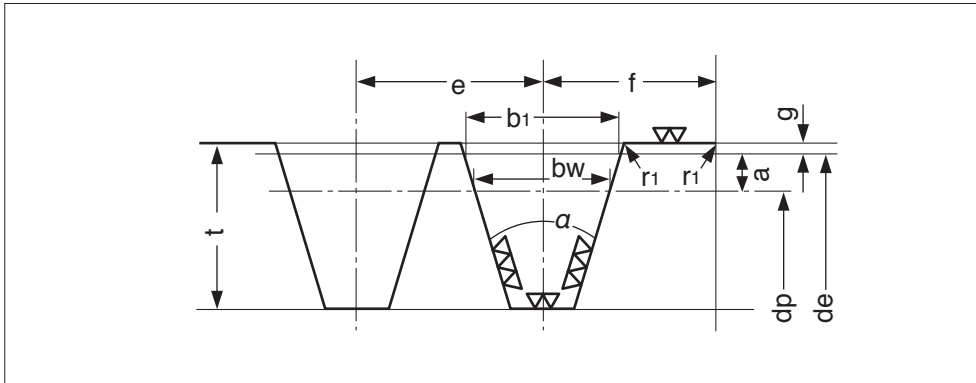
Number of belts & pulley width b2 (mm)

Table 1-15

Number of belt	Z, ZX SPZ, SPZX	A, AX SPA, SPAX	B, BX SPB, SPBX	C, CX SPC, SPCX	D	E
1	16	20	25	34	48	58
2	28	35	44	59.5	85	102.5
3	40	50	63	85	122	147
4	52	65	82	110.5	159	191.5
5	64	80	101	136	193	236
6	76	95	120	161.5	233	280.5
7	88	110	139	187	270	325
8	100	125	158	212.5	307	369.5
9	112	140	177	238	344	414
10	124	155	196	263.5	381	458.5
11	136	170	215	289	418	503
12	148	185	234	314.5	455	547.5

Pulley for Maxstar Wedge V-Belts for RMA / MPTA

Fig. 1-17



Dimensions(mm)

Table 1-16

Belt section	Effective diameter: d_e	α (°)	Effective width b_1	Pitch width b_w	a	Groove depth min. t	Pitch e	Groove spacing: f	g
3V (3VX)	≤ 90	36 ± 0.5	8.9 ± 0.13	8.51	0.6	8.9	10.3 ± 0.25	8.7	0.20
	$90 < d_e \leq 150$	38 ± 0.5		8.49					
	$150 < d_e \leq 300$	40 ± 0.5		8.46					
	> 300	42 ± 0.5		8.44					
5V (5VX)	≤ 250	38 ± 0.5	15.2 ± 0.13	14.30	1.3	15.2	17.5 ± 0.25	12.7	0.25
	$250 < d_e \leq 400$	40 ± 0.5		14.25					
	> 400	42 ± 0.5		14.20					
8V	≤ 400	38 ± 0.5	25.4 ± 0.13	23.68	2.5	25.4	28.6 ± 0.25	19.0	0.30
	$400 < d_e \leq 560$	40 ± 0.5		23.58					
	> 560	42 ± 0.5		23.48					

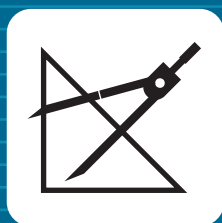
Number of belts & pulley width(mm)

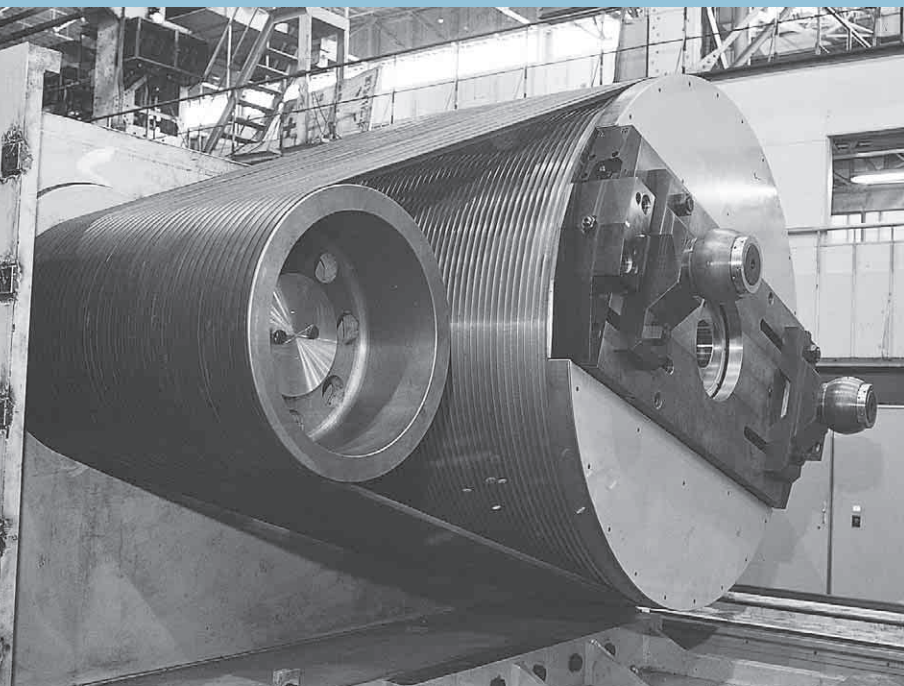
Table 1-17

Belt Section \ Number of belts	1	2	3	4	5	6	7	8	9	10
3V	18	28	38	49	59	69	80	90	100	111
5V	26	43	61	78	96	113	131	148	166	183
8V	38	67	96	124	153	181	210	239	267	296



Design





2.Design

Design process

Calculation example

Formulas for V-Belt drive design

Power Rating Table

Drive Selection Table

Design process for Classical V-Belts / Narrow V-Belts

Design Flow

1

Set conditions required in design work.

a. Type of machine

b. Transmission power

It is ideal to use the actual load applied to the belt as the value of the transmission power, but the rated power of the motor is commonly used for calculation.

c. Running hours in a single day

d. Small pulley speed

e. Speed ratio

$$\text{Speed ratio} = \frac{\text{Large pulley datum diameter}}{\text{Small pulley datum diameter}}$$

f. Interim center distance

g. Special uses and environmental conditions

Contact us for the case of exposure to high or low temperature, water, oil, acid or alkali.

Design Flow

2

Set the design power.

1. How to calculate the service factor (Ks)

$$K_s = K_o + K_i + K_e$$

Wherein, Ks : Service factor

Ko : Service correction factor >> (Table 2-1)

Ki : Idler correction factor >> (Table 2-2)

Ke : Environment correction factor >> (Table 2-3)

2. How to calculate the design power (Pd)

$$P_d = P_t \times K_s$$

Wherein, Pd : Design power (kW)

Pt : Transmission power (kW)

Ks : Service factor

The value of transmission power used in designing is the power requirement of the driven machine, if obtained, or the power of driving unit (engine or motor).

Convert the value from torque (Tq) or horse power (PS) into kW with the formula below.

$$P_t = \frac{T_q \times n}{9.55 \times 10^3}$$

Wherein, Pt : Transmission power (kW)

Tq : Torque (N·m)

n : Shaft speed (rpm)

$$1 \text{ PS} = 0.7355 \text{ kW}$$

1. Ko

Service correction factor (Ko)

Table 2-1

Driven Machine	Driving unit / Motor					
	Max power $\leq$ 300% of rated power			Max power > 300% of rated power		
	AC motors, single-and three-phase with star-delta start. DC shunt-wound motors, Multiple cylinder internal combustion engines.			AC motors, single and three-phase, series wound, slip-ring motors with direct start. DC motors, series and compound wound. Single cylinder internal combustion engines.		
	Running time (hrs./day)			Running time (hrs./day)		
	3 ~ 5	8 ~ 12	16 ~ 24	3 ~ 5	8 ~ 12	16 ~ 24
● Agitator for liquid ● Small centrifugal blower ● Fan up to 7.5 kW ● Light-duty conveyor	1.0	1.1	1.2	1.1	1.2	1.3
● Belt conveyor (for sand, grain, etc.) ● Dough mixer ● Fan over 7.5 kW ● Generator ● Machine tool ● Punching machine ● Pressing machine ● Shearing machine ● Printing machine ● Positive displacement rotary pump ● Vibrating and rotary screen	1.1	1.2	1.3	1.2	1.3	1.4
● Brick-making machinery ● Bucket elevator ● Piston compressor ● Screw conveyor ● Hammer mill ● Hollander ● Piston pump ● Positive displacement blower ● Crusher ● Woodworking machinery ● Textile machinery	1.2	1.3	1.4	1.4	1.5	1.6
● Gyratory and jaw-roll crusher ● Mill (ball/rod) ● Hoist (heavy load) ● Rolling mill, calender etc, for the rubber and plastic industry	1.3	1.4	1.5	1.5	1.6	1.8

2. Ki

Idler correction factor (Ki)

Table 2-2

Location of Idler	Ki
Belt slack side, inside of belt	0.0
Belt slack side, outside of belt	0.1
Belt tight side, inside of belt	0.1
Belt tight side, outside of belt	0.2

3. Ke

Environment correction factor (Ke)

Table 2-3

Environmental condition	Ke
Frequent start and stop of machine	0.2
Hard to conduct maintenance checkup	0.2
Dusty environment	0.2
High temperature	0.2
Oil or water splashing	0.2

● Avoid oil and water splash by cover to prevent belt slipping.



3

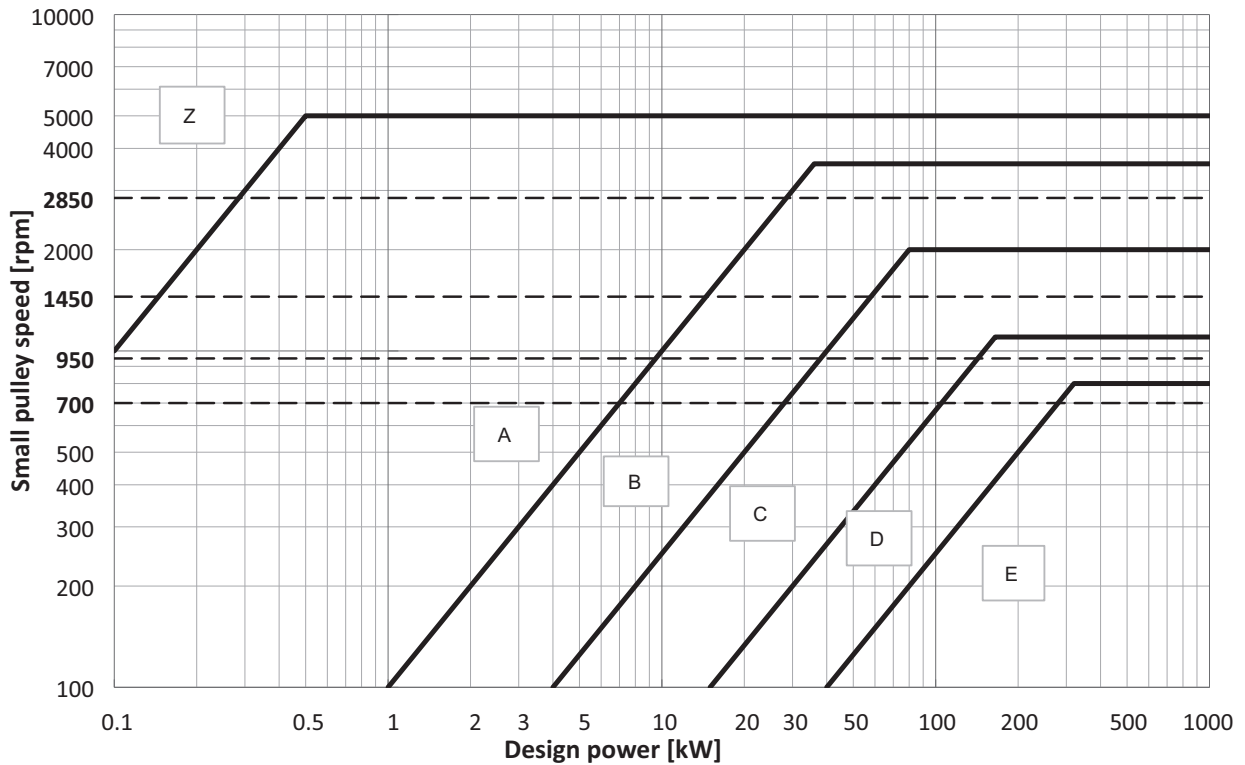
Select the belt type.

● Select the belt type in the selection charts below according to design power and small pulley speed.

● If the intersection locates near the dividing line, select belt type considering other conditions such as pulley cost.

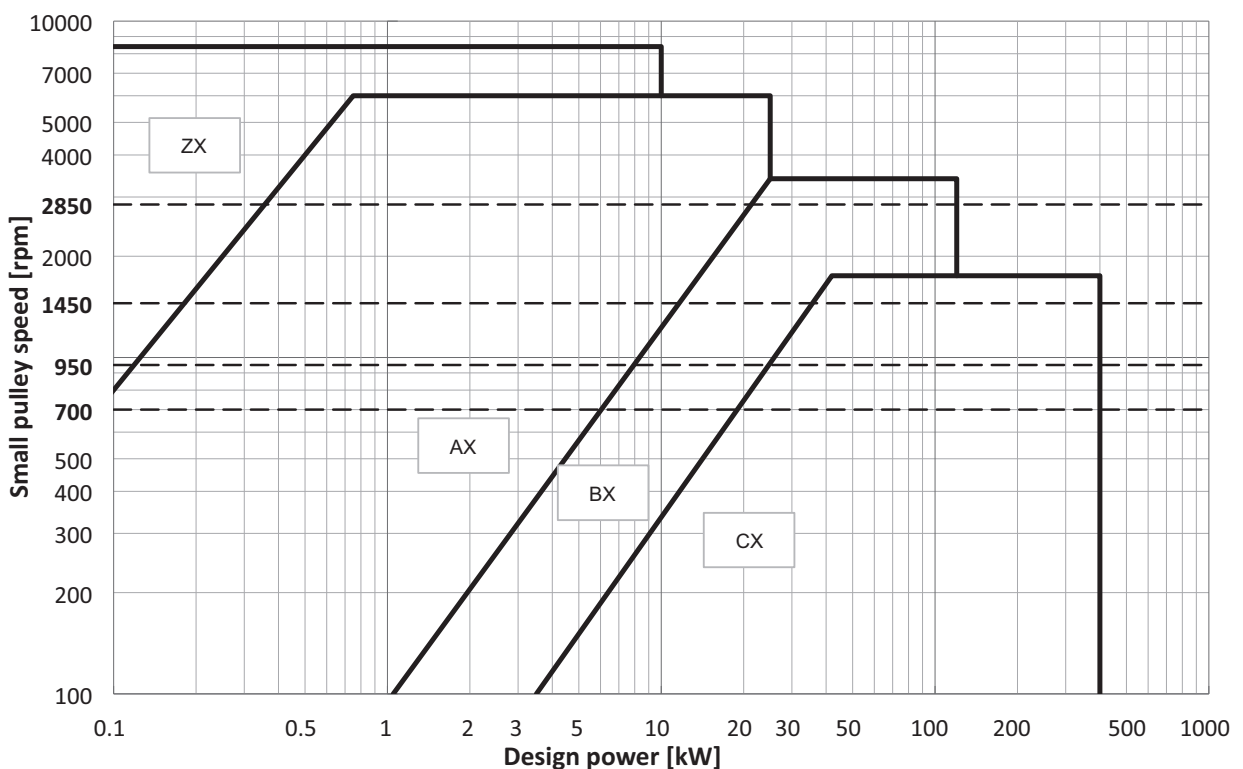
Cross section selection chart for Classical V-Belts for DIN

Fig. 2-1



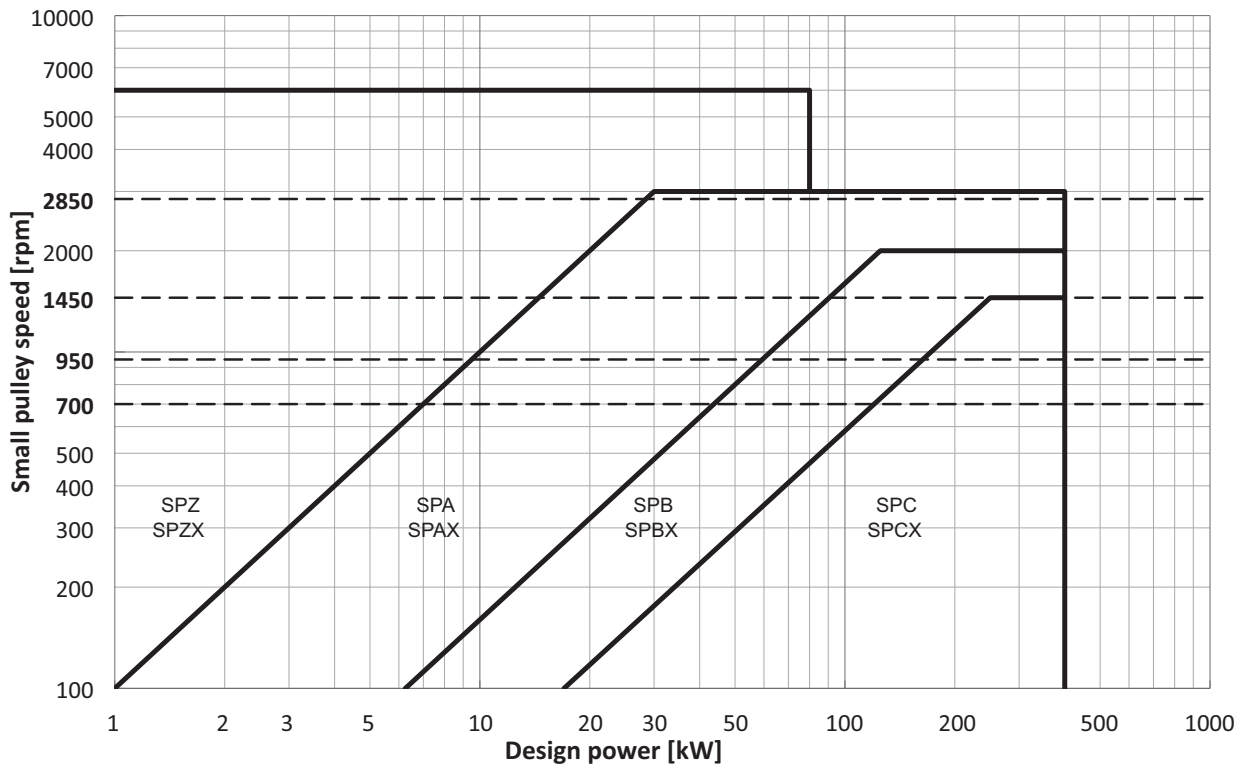
Cross section selection chart for Classical Raw Edge Cogged V-Belts

Fig. 2-2



Cross section selection chart for Narrow V-Belts for DIN

Fig. 2-3



Design Flow

4

Select the pulley size.

- Select the small pulley datum diameter larger than the minimum specified in the Table 2-4. Inappropriate pulley reduces transmission efficiency and belts' durability significantly.

Minimum pulley datum diameter	Unit : mm						Table 2-4
Belt type	Z	A	B	C	D	E	
For wrapped type	45	71	112	180	315	450	
For raw edge cogged type	40	63	90	140	-	-	

- Calculate the large pulley datum diameter.

$$D_d = d_d \times SR$$

D_d : Large pulley datum diameter (mm)
 d_d : Small pulley datum diameter (mm)
 SR : Speed ratio

Difference between pulley outside diameter and datum diameter is specified in the following table.

Difference between pulley outside diameter and datum diameter	Unit : mm						Table 2-5
Belt type	Z	A	B	C	D	E	
Difference	5.0	6.6	8.4	11.4	16.2	24.0	

Design Flow

5

Determine the belt length.

- 1) Determine the interim belt datum length.

$$L_d' = 2C' + 1.57(D_d + d_d)$$

L_d' : Interim belt datum length (mm)
 C' : Interim center distance (mm)
 D_d : Large pulley datum diameter (mm)
 d_d : Small pulley datum diameter (mm)

- 2) Select the standard belt length closest to the L_d' from our lineup.

Center distance is calculated from the following formula.

$$C = \frac{b + \sqrt{b^2 - 8(D_d - d_d)^2}}{8}$$

C : Center distance (mm)
 $b = 2L_d - \pi(D_d + d_d)$
 L_d : Belt datum length (mm)

- 3) If center distance is predetermined, use the following formula to determine interim belt datum length.

$$L_d' = 2C + \frac{\pi}{2}(D_d + d_d) + \frac{(D_d - d_d)^2}{4C}$$

2
Design



6

Determine the required number of belts.

Required number of belts (nb) is determined as follows.
Round up the calculation results.

$$nb = \frac{P_d}{P_c}$$

$$\uparrow P_c = (P_s + P_a) \times K_c$$

$$\uparrow K_c = K_\theta \times K_\ell$$

$$\uparrow K_\ell = \frac{D_d - d_d}{C}$$

nb : Required number of belts
 Pd : Design power (kW)
 Pc : Correction power rating (kW)
 Ps : Basic power rating (kW)
 Pa : Additional power rating for speed ratio (kW)
 Kc : Power rating correction factor
 Kθ : Arc of contact correction factor
 Kℓ : Belt length correction factor
 Dd : Large pulley datum diameter (mm)
 dd : Small pulley datum diameter (mm)
 C : Center distance (mm)

● Arc of contact correction factor : Kθ Table 2-6

$\frac{D_d - d_d}{C}$	Contact angle on small pulley $\theta(^{\circ})$	Kθ
0.00	180	1.00
0.10	174	0.99
0.20	169	0.97
0.30	163	0.96
0.40	157	0.94
0.50	151	0.93
0.60	145	0.91
0.70	139	0.89
0.80	133	0.87
0.90	127	0.85
1.00	120	0.82
1.10	113	0.80
1.20	106	0.77
1.30	99	0.73
1.40	91	0.70
1.50	83	0.65

Contact angle on small pulley : $\theta = 180 - 2\sin^{-1} \frac{D_d - d_d}{2C}$

Contact angle on large pulley : $\theta = 180 + 2\sin^{-1} \frac{D_d - d_d}{2C}$

Dd : Large pulley datum diameter (mm)

dd : Small pulley datum diameter (mm)

C : Center distance (mm)

● Belt length correction factor for Classical V-Belts: Kℓ Table 2-7

Length designation	A	B	C	D	E
20 ~ 25	0.77	0.72			
26 ~ 30	0.82	0.76			
31 ~ 34	0.85	0.79			
35 ~ 37	0.87	0.81	0.71		
38 ~ 41	0.89	0.83	0.73		
42 ~ 45	0.91	0.85	0.75		
46 ~ 50	0.93	0.87	0.77		
51 ~ 54	0.94	0.89	0.78		
55 ~ 59	0.96	0.91	0.80		
60 ~ 67	0.98	0.93	0.82		
68 ~ 74	1.01	0.95	0.84		
75 ~ 79	1.03	0.97	0.86		
80 ~ 84	1.04	0.98	0.87		
85 ~ 89	1.05	0.99	0.89		
90 ~ 95	1.07	1.01	0.90		
96 ~ 104	1.08	1.03	0.91	0.81	
105 ~ 111	1.10	1.04	0.93	0.82	
112 ~ 119	1.12	1.06	0.94	0.84	
120 ~ 127	1.13	1.07	0.96	0.85	
128 ~ 144	1.15	1.09	0.98	0.87	0.85
145 ~ 154	1.18	1.11	1.00	0.89	0.87
155 ~ 169	1.19	1.13	1.02	0.91	0.88
170 ~ 179	1.21	1.15	1.03	0.92	0.90
180 ~ 194	1.23	1.17	1.05	0.94	0.91
195 ~ 209	1.25	1.18	1.07	0.95	0.93
210 ~ 239	1.27	1.21	1.09	0.98	0.95
240 ~ 269	1.30	1.24	1.12	1.00	0.98
270 ~ 299	1.33	1.26	1.14	1.03	1.00
300 ~ 329	1.35	1.29	1.17	1.05	1.02
330 ~ 359	1.38	1.31	1.19	1.07	1.04
360 ~ 389	1.40	1.33	1.21	1.09	1.06
390 ~ 419		1.35	1.22	1.11	1.08
420 ~ 479		1.38	1.25	1.13	1.10
480 ~ 539		1.41	1.28	1.16	1.13
540 ~ 600		1.44	1.31	1.18	1.16
601 ~ 660		1.46	1.33	1.21	1.18

● Belt length correction factor for Narrow V-Belts: Kℓ Table 2-8

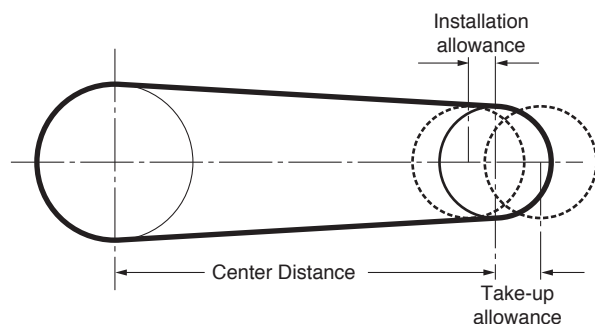
Length designation	Belt length correction factor : Kℓ			
	SPZ SPZX	SPA SPAX	SPB SPBX	SPC SPCX
487 ~ 670	0.80			
671 ~ 755	0.84	0.79		
756 ~ 850	0.86	0.81		
851 ~ 950	0.89	0.83		
951 ~ 1060	0.91	0.85		
1061 ~ 1185	0.93	0.87		
1186 ~ 1325	0.95	0.89	0.82	
1326 ~ 1500	0.98	0.91	0.84	
1501 ~ 1700	1.00	0.93	0.86	
1701 ~ 1900	1.02	0.95	0.88	
1901 ~ 2120	1.05	0.96	0.90	0.82
2121 ~ 2370	1.07	0.98	0.92	0.84
2371 ~ 2650	1.09	1.00	0.94	0.86
2651 ~ 2975	1.11	1.02	0.96	0.88
2976 ~ 3350	1.13	1.04	0.98	0.90
3351 ~ 3775	1.16	1.06	1.00	0.92
3776 ~ 4250	1.18	1.08	1.02	0.94
4251 ~ 4750	1.20	1.10	1.04	0.96
4751 ~ 5300		1.12	1.06	0.98
5301 ~ 5950			1.08	1.00
5951 ~ 6700			1.10	1.02
6701 ~ 7550			1.12	1.04
7551 ~ 8500			1.14	1.06
8501 ~ 9500			1.16	1.08
9501 ~ 10000			1.17	1.10



Installation and take-up allowance.

Installation and take-up allowance are as follows.
Use idler pulley if you cannot arrange allowance.

Fig. 2-4



● Installation and take-up allowance for Classical V-Belts

Table 2-9

Length designation	Datum length (mm)	Installation allowance (mm)						Take-up allowance (mm)
		Z	A	B	C	D	E	All sections
20 ~ 38	$500 \leq L_d < 970$	15	20	25				25
39 ~ 60	$970 \leq L_d < 1500$	20	20	25	40			40
61 ~ 90	$1500 \leq L_d < 2200$		20	35	40			50
91 ~ 120	$2200 \leq L_d < 3000$		25	35	40			65
121 ~ 158	$3000 \leq L_d < 4000$		25	35	40	50		75
159 ~ 195	$4000 \leq L_d < 5000$			35	50	50	65	90
196 ~ 240	$5000 \leq L_d < 6000$			40	50	50	65	100
241 ~ 270	$6000 \leq L_d < 6800$				50	65	65	115
271 ~ 330	$6800 \leq L_d < 8400$				50	65	75	130
331 ~ 400	$8400 \leq L_d < 10000$				50	65	75	155
400 ~	$10000 \leq L_d$					75	90	$L_d \times 0.015$

● Installation and take-up allowance for Narrow V-Belts

Table 2-10

Length designation	Installation allowance (mm)				Take-up allowance (mm)
	SPZ SPZX	SPA SPAX	SPB SPBX	SPC SPCX	All sections
487 ~ 670	16	19			10
670 ~ 1000	18	21			10
1000 ~ 1320	19	22	27		13
1320 ~ 1662	21	24	29		17
1662 ~ 2240	24	27	32	39	22
2240 ~ 3000	28	31	35	43	30
3000 ~ 3550	30	33	38	46	36
3550 ~ 4500	35	38	43	51	45
4500 ~ 5600	41	44	48	56	56
5600 ~ 6700	46		54	62	67
6700 ~ 8500	55		63	71	85
8500 ~ 10000			70	78	100



Calculation example for Classical V-Belts / Narrow V-Belts

Design Flow

1

Set conditions required in design work.

- a. Type of machine ... Compressor
- b. Transmission power ... Four pole motor 3.75kW/1750rpm
- c. Running hours in a single day ... 8 hours / day
- d. Small pulley speed ... 1750rpm

- e. Speed ratio ... 2 : 1 (Deceleration)
- f. Interim center distance ... 300mm
- g. Special uses and environmental conditions ... None

Design Flow

2

Set the design power.

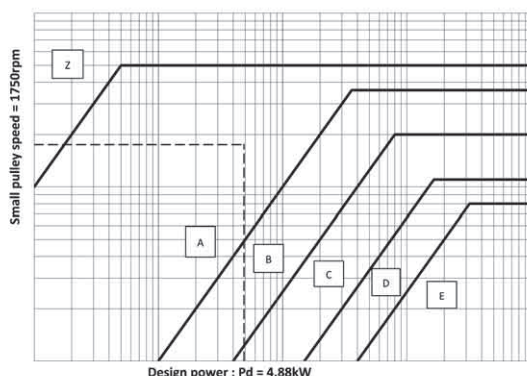
Service correction factor : $K_o = 1.3$ (Table 2-1)
 Idler correction factor : $K_i = 0$ (Table 2-2)
 Environment correction factor : $K_e = 0$ (Table 2-3)
 Service factor : $K_s = K_o + K_i + K_e$
 $= 1.3 + 0 + 0$
 $= 1.3$
 Design power : $P_d = P_t \times K_s$
 $= 3.75 \times 1.3$
 $= 4.88 \text{ kW}$

Design Flow

3

Select the belt type.

Select the belt type in Cross section selection chart.
 The lines of P_d (4.88kW) and small pulley speed (1750rpm) intersect in A section.



Design Flow

4

Select the pulley size.

- 1) Select the small pulley of 95mm datum diameter.
 (The recommended minimum pulley datum diameter is specified in Table 2-4.)
- 2) Calculate the large pulley datum diameter.

$$\begin{aligned} \text{Large pulley datum diameter : } D_d &= d_d \times SR \\ &= 95 \times 2 \\ &= 190 \text{ mm} \end{aligned}$$

Design Flow

5

Determine the belt length.

- 1) Determine the interim belt datum length.

$$L_d' = 2 \times 300 + 1.57 \times (190 + 95) = 1047.5 \text{ mm}$$

- 2) Select the standard belt length closest to the L_d' from Table 1-2.

Belt datum length 1046mm : Length designation A40 is selected.

Center distance is calculated as follows.

$$\begin{aligned} b &= 2 \times 1046 - 3.14 \times (190 + 95) = 1197 \\ C &= \frac{1197 + \sqrt{1197^2 - 8 \times (190 - 95)^2}}{8} = 295.4 \\ \text{Center distance : } C &= 295 \text{ mm} \end{aligned}$$

Design Flow

6

Determine the required number of belts.

$$\begin{aligned} P_s &= 2.16 \text{ kW (Refer to Power Rating Table)} \\ P_a &= 0.37 \text{ kW (Refer to Power Rating Table)} \\ \frac{D_d - d_d}{C} &= \frac{190 - 95}{295} = 0.32 \\ K_\theta &= 0.96 \text{ (Refer to Table 2-6).} \\ K_l &= 0.89 \text{ (Refer to Table 2-7).} \\ K_c &= K_\theta \times K_l = 0.96 \times 0.89 = 0.85 \\ P_c &= (P_s + P_a) \times K_c = (2.16 + 0.37) \times 0.85 = 2.15 \text{ kW} \\ n_b &= \frac{P_d}{P_c} = \frac{4.88}{2.15} = 2.3 \rightarrow 3 \text{ pcs.} \end{aligned}$$

Design Flow

7

Installation and take-up allowance.

Installation and take-up allowance are obtained from Table 2-9.

$$\begin{aligned} \text{Installation allowance} &= 20 \text{ mm} \\ &\rightarrow \text{Minimum center distance} = 295 - 20 = 275 \text{ mm} \\ \text{Take-up allowance} &= 40 \text{ mm} \\ &\rightarrow \text{Maximum center distance} = 295 + 40 = 335 \text{ mm} \end{aligned}$$

Design process for Maxstar Wedge V-Belts

Design Flow

1

Set conditions required in design work.

a. Type of machine

b. Transmission power

It is ideal to use the actual load applied to the belt as the value of the transmission power, but the rated power of the motor is commonly used for calculation.

c. Running hours in a single day

d. Small pulley speed

e. Speed ratio

$$\text{Speed ratio} = \frac{\text{Large pulley pitch diameter}}{\text{Small pulley pitch diameter}}$$

f. Interim center distance

g. Special uses and environmental conditions

Contact us for the case of exposure to high or low temperature, water, oil, acid or alkali.

Design Flow

2

Set the design power.

1. How to calculate the service factor (Ks)

$$K_s = K_o + K_i + K_e$$

Wherein, Ks : Service factor

Ko : Service correction factor >> (Table 2-11)

Ki : Idler correction factor >> (Table 2-12)

Ke : Environment correction factor >> (Table 2-13)

2. How to calculate the design power (Pd)

$$P_d = P_t \times K_s$$

Wherein, Pd : Design power (kW)

Pt : Transmission power (kW)

Ks : Service factor

The value of transmission power used in designing is the power requirement of the driven machine, if obtained, or the power of driving unit (engine or motor).

Convert the value from torque (Tq) or horse power (PS) into kW with the formula below.

$$P_t = \frac{T_q \times n}{9.55 \times 10^3}$$

Wherein, Pt : Transmission power (kW)

Tq : Torque (N·m)

n : Shaft speed (rpm)

$$1 \text{ PS} = 0.7355 \text{ kW}$$

2
Design



1. Ko

Service correction factor (Ko)

Table 2-11

Driven Machine	Driving unit / Motor					
	Max power $\leq$ 300% of rated power			Max power > 300% of rated power		
	AC motors, single-and three-phase with star-delta start. DC shunt-wound motors, Multiple cylinder internal combustion engines.			AC motors, single and three-phase, series wound, slip-ring motors with direct start. DC motors, series and compound wound. Single cylinder internal combustion engines.		
	Running time (hrs./day)			Running time (hrs./day)		
	3 ~ 5	8 ~ 12	16 ~ 24	3 ~ 5	8 ~ 12	16 ~ 24
● Agitator for liquid ● Small centrifugal blower ● Fan up to 7.5 kW ● Light-duty conveyor	1.0	1.1	1.2	1.1	1.2	1.3
● Belt conveyor (for sand, grain, etc.) ● Dough mixer ● Fan over 7.5 kW ● Generator ● Machine tool ● Punching machine ● Pressing machine ● Shearing machine ● Printing machine ● Positive displacement rotary pump ● Vibrating and rotary screen	1.1	1.2	1.3	1.2	1.3	1.4
● Brick-making machinery ● Bucket elevator ● Piston compressor ● Screw conveyor ● Hammer mill ● Hollander ● Piston pump ● Positive displacement blower ● Crusher ● Woodworking machinery ● Textile machinery	1.2	1.3	1.4	1.4	1.5	1.6
● Gyratory and jaw-roll crusher ● Mill (ball/rod) ● Hoist (heavy load) ● Rolling mill, calender etc, for the rubber and plastic industry	1.3	1.4	1.5	1.5	1.6	1.8

2. Ki

Idler correction factor (Ki)

Table 2-12

Location of Idler	Ki
Belt slack side, inside of belt	0.0
Belt slack side, outside of belt	0.1
Belt tight side, inside of belt	0.1
Belt tight side, outside of belt	0.2

3. Ke

Environment correction factor (Ke)

Table 2-13

Environmental condition	Ke
Frequent start and stop of machine	0.2
Hard to conduct maintenance checkup	0.2
Dusty environment	0.2
High temperature	0.2
Oil or water splashing	0.2

● Avoid oil and water splash by cover to prevent belt slipping.



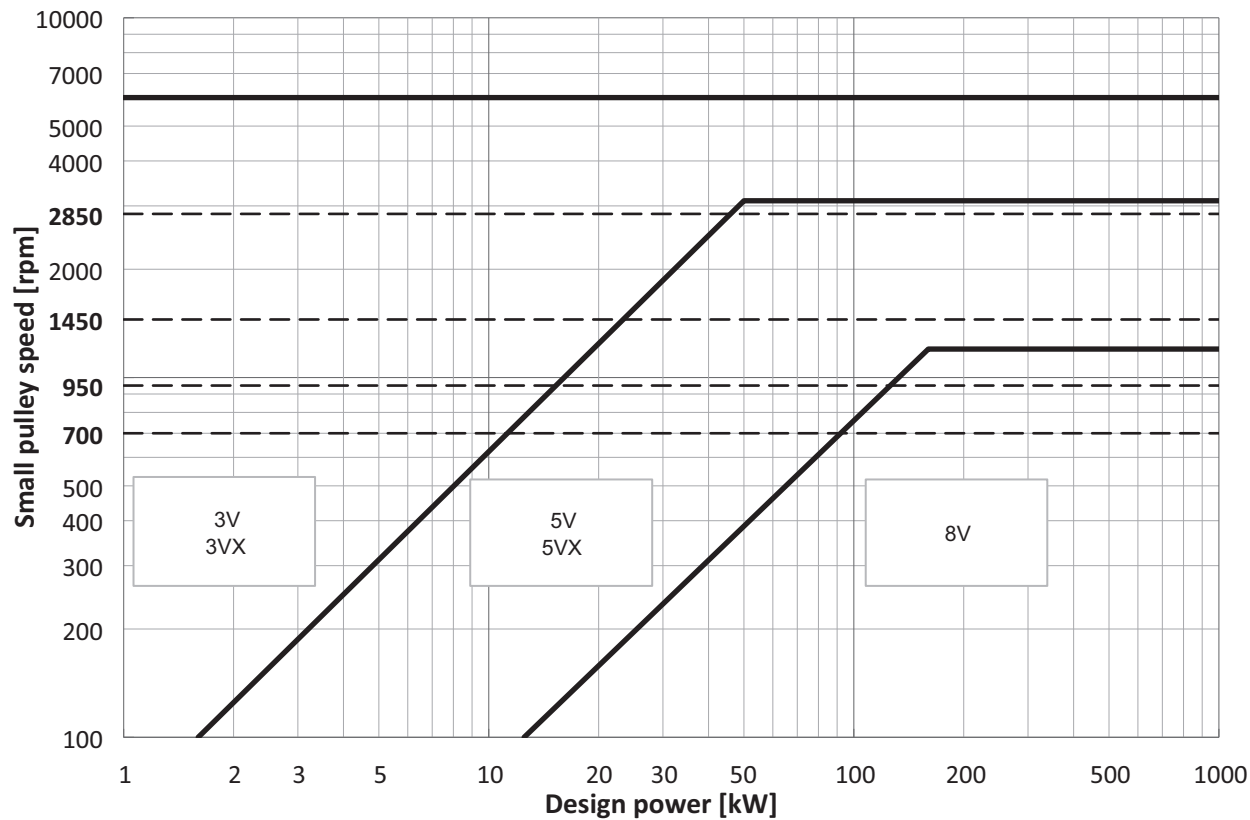
3

Select the belt type.

- Select the belt type in the selection charts below according to design power and small pulley speed.
- If the intersection locates near the dividing line, select belt type considering other conditions such as pulley cost.

Cross section selection chart for Maxstar Wedge V-Belts

Fig. 2-5



4

Select the pulley size.

4-A : Select from the standard pulleys.

You can obtain pulley size, belt length, and center distance easily by using Drive Selection Table on page 2-43 ~ 2-62.

Speed ratio	Effective diameter (mm)		Center distance (mm)							
	Small pulley	Large pulley	3V 250	3V 265	3V 280	3V 300	3V 315	3V 335	3V 355	4)
1.48	85	125	151	170	190	215	234	260	285	
1.48	95	140	131	150	169	195	214	240	265	
1.50	67	100	186	205	224	249	268	294	319	
1.50	75	112	170	189	208	233	252	278	304	
1.51	100	150	-	138	157	183	202	228	253	
1.56	90	140	135	154	173	193	213	244	269	3)
1.57	80	125	155	174	193	219	238	264	289	
1.57	160	250	-	-	-	-	-	-	-	
1.58	200	315	-	-	-	-	-	-	-	
1.59	71	112	173	192	211	236	255	281	307	

- 1) Choose the speed ratio to satisfy your drive design.
- 2) Choose the pulley size.
- 3) Choose the center distance.
- 4) Look up to find the belt length.
→Go to design flow ⑥

4-B : With the nonstandard pulley.

Follow the procedure below if you cannot find the desired speed ratio in the Drive Selection Table.

- 1) Choose the speed ratio closest to the desired speed ratio.
- 2) Choose the large pulley from the standard pulleys.
- 3) Calculate the effective diameter of the small pulley from the following formula.

$$de = \frac{D_p}{SR} + (\text{Difference between effective diameter and pitch diameter})$$

de : Small pulley effective diameter
Dp : Large pulley pitch diameter
SR : Speed ratio

Difference between effective diameter and pitch diameter Table 2-14

Belt type	3V-3VX	5V-5VX	8V
Difference (mm)	1.2	2.6	5.0

Pulley diameter should be larger than the minimum pulley effective diameter specified in Table 2-15.

Minimum pulley effective diameter Table 2-15

Belt type	3V	3VX	5V	5VX	8V
Minimum pulley effective diameter (mm)	67	56	180	112	315

5

Determine the belt length.

5-A : Determine from Drive Selection Table.

- 1) Choose the speed ratio and center distance closest to your drive in Drive Selection Table on page 2-43 ~ 2-62.
- 2) Precise center distance is calculated by adding the following correction value to the center distance in Drive Selection Table.

Center distance correction value = $0.8 \times (\text{Effective diameter of the standard small pulley} - \text{Effective diameter of the calculated small pulley})$

5-B : Determine from the calculation formula.

- 1) Determine the interim belt effective length.

$$Le' = 2C' + 1.57(De + de)$$

Le' : Interim belt effective length (mm)

C' : Interim center distance (mm)

De : Large pulley effective diameter (mm)

de : Small pulley effective diameter (mm)

- 2) Select the standard belt length closest to the Lp' from our lineup.

Center distance is calculated by the following formula.

$$C = \frac{b + \sqrt{b^2 - 8(De - de)^2}}{8}$$

C : Center distance (mm)

b : $2Le - \pi(De + de)$

Le : Belt effective length (mm)



6

Determine the required number of belts.

Required number of belts (nb) is determined as follows.
Round up the calculation results.

$$nb = \frac{Pd}{Pc}$$

↑ $Pc = (Ps + Pa) \times Kc$

↑ $Kc = K\theta \times K\ell$

↑ $\frac{De - de}{C}$

nb : Required number of belt
Pd : Design power (kW)
Pc : Correction power rating (kW)
Ps : Basic power rating (kW)
Pa : Additional power rating for speed ratio (kW)
Kc : Power rating correction factor
Kθ : Arc of contact correction factor
Kℓ : Belt length correction factor
De : Large pulley effective diameter (mm)
de : Small pulley effective diameter (mm)
C : Center distance (mm)

With the standard pulley, you can obtain Kc easily from Drive Selection Table.

Kc is as in Color coding below.

Color coding of Power rating correction factor : Kc

0.7	0.8	0.9	1.0	1.1	1.2
-----	-----	-----	-----	-----	-----

● Arc of contact correction factor : Kθ Table 2-16

$\frac{De-de}{C}$	Contact angle on small pulley $\theta(^{\circ})$	Kθ
0.00	180	1.00
0.10	174	0.99
0.20	169	0.97
0.30	163	0.96
0.40	157	0.94
0.50	151	0.93
0.60	145	0.91
0.70	139	0.89
0.80	133	0.87
0.90	127	0.85
1.00	120	0.82
1.10	113	0.80
1.20	106	0.77
1.30	99	0.73
1.40	91	0.70
1.50	83	0.65

Contact angle on small pulley : $\theta = 180 - 2\sin^{-1} \frac{De - de}{2C}$

Contact angle on large pulley : $\theta = 180 + 2\sin^{-1} \frac{De - de}{2C}$

De : Large pulley effective diameter (mm)
de : Small pulley effective diameter (mm)
C : Center distance (mm)

● Belt length correction factor for Maxstar
Wedge V-Belts: Kℓ

Table 2-17

Length designation	Belt length correction factor : Kℓ		
	3V 3VX	5V 5VX	8V
250	0.83		
265	0.84		
280	0.85		
300	0.86		
315	0.87		
335	0.88		
355	0.89		
375	0.90		
400	0.92		
425	0.93		
450	0.94		
475	0.95		
500	0.96	0.85	
530	0.97	0.86	
560	0.98	0.87	
600	0.99	0.88	
630	1.00	0.89	
670	1.01	0.90	
710	1.02	0.91	
750	1.03	0.92	
800	1.04	0.93	
850	1.06	0.94	
900	1.07	0.95	
950	1.08	0.96	
1000	1.09	0.96	0.87
1060	1.10	0.97	0.88
1120	1.11	0.98	0.88
1180	1.12	0.99	0.89
1250	1.13	1.00	0.90
1320	1.14	1.01	0.91
1400	1.15	1.02	0.92
1500		1.03	0.93
1600		1.04	0.94
1700		1.05	0.94
1800		1.06	0.95
1900		1.07	0.96
2000		1.08	0.97
2120		1.09	0.98
2240		1.09	0.98
2360		1.10	0.99
2500		1.11	1.00
2650		1.12	1.01
2800		1.13	1.02
3000		1.14	1.03
3150		1.15	1.03
3350		1.16	1.04
3550		1.17	1.05
3750			1.06
4000			1.07
4250			1.08
4500			1.09
4750			1.09
5000			1.10
5600			1.11
6000			1.13

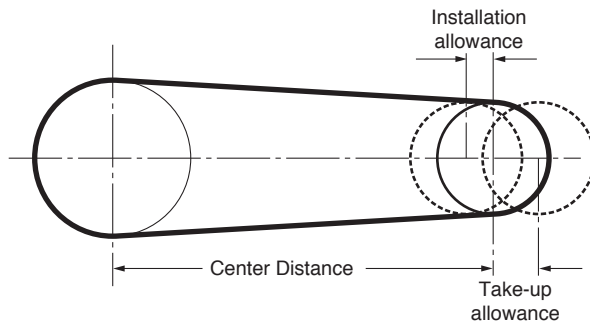




Installation and take-up allowance.

Installation and take-up allowance are as follows.
Use idler pulley if you cannot arrange allowance.

Fig. 2-6



● Installation and take-up allowance for Maxstar Wedge V-Belts

Table 2-18

Length designation	Installation allowance (mm)	Take-up allowance (mm)	Sum of installation and take-up allowance (mm)
3V250 ~ 3V475	15	25	40
3V500 ~ 3V710	20	35	55
3V750 ~ 3V1060	20	40	60
3V1120 ~ 3V1250	20	50	70
3V1320, 3V1400	20	60	80
5V500 ~ 5V710	25	35	60
5V750 ~ 5V1060	25	40	65
5V1120 ~ 5V1250	25	50	75
5V1320 ~ 5V1700	25	60	85
5V1800 ~ 5V2000	25	65	90
5V2120 ~ 5V2240	35	75	110
5V2360	35	80	115
5V2500, 5V2560	35	85	120
5V2800, 5V3000	35	90	125
5V3150, 5V3550	35	105	140
8V1000, 8V1060	40	40	80
8V1120 ~ 8V1250	40	50	90
8V1320 ~ 8V1700	40	60	100
8V1800 ~ 8V2000	50	65	115
8V2120 ~ 8V2240	50	75	125
8V2360	50	80	130
8V2500, 8V2650	50	85	135
8V2800, 8V3000	50	90	140
8V3150	50	105	155
8V3350, 8V3550	55	105	160
8V3750	55	115	170
8V4000 ~ 8V5600	55	140	195

Calculation example for Maxstar Wedge V-Belts #1 (With the standard pulley)

Design Flow

1

Set conditions required in design work.

- a. Type of machine ... Blower
- b. Transmission power ... 3.7kW
- c. Running hours in a single day ... 8 hours / day
- d. Small pulley speed ... 1750rpm

- e. Speed ratio ... 1.87 (Deceleration)
- f. Interim center distance ... 400mm
- g. Special uses and environmental conditions ... None

Design Flow

2

Set the design power.

Service correction factor : $K_o = 1.1$ (Table 2-11)
 Idler correction factor : $K_i = 0$ (Table 2-12)
 Environment correction factor : $K_e = 0$ (Table 2-13)
 Service factor : $K_s = K_o + K_i + K_e$
 $= 1.1 + 0 + 0$
 $= 1.1$
 Design power : $P_d = P_t \times K_s$
 $= 3.7 \times 1.1$
 $= 4.07 \text{ kW}$

Design Flow

6

Determine the required number of belts.

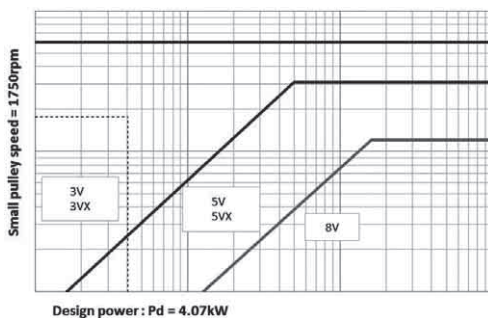
$P_s = 1.69 \text{ kW}$ (Refer to Power Rating Table)
 $P_a = 0.26 \text{ kW}$ (Refer to Power Rating Table)
 $K_c = 0.9$ (Refer to Drive Selection Table)
 $P_c = (P_s + P_a) \times K_c = (1.69 + 0.26) \times 0.9 = 1.76 \text{ kW}$
 $n_b = \frac{P_d}{P_c} = \frac{4.07}{1.76} = 2.3 \rightarrow 3 \text{ pcs.}$

Design Flow

3

Select the belt type.

Select the belt type in Cross section selection chart.
 The lines of P_d (4.07kW) and small pulley speed (1750rpm) intersect in 3V section.



Design Flow

7

Installation and take-up allowance.

Installation and take-up allowance are obtained from Table 2-18.

Installation allowance = 15 mm
 $\rightarrow$ Minimum center distance = $401 - 15 = 386 \text{ mm}$
 Take-up allowance = 25 mm
 $\rightarrow$ Maximum center distance = $401 + 25 = 426 \text{ mm}$

Summary

Belt : 3V-450, 3 pcs.

Driver pulley : 75-3V-3 (Effective diameter = 75mm, 3 grooves)

Driven pulley : 140-3V-3 (Effective diameter = 140mm, 3 grooves)

Center distance : 401mm (+25mm / -15mm)

Design Flow

4

5

Select the pulley size.
 Determine the belt length.

Speed ratio	Effective diameter (mm)		Center distance (mm)							
	Small pulley	Large pulley	3V 250	3V 265	3V 355	3V 375	3V 400	3V 425	3V 450	4)
1.88 1)	67	125 2)	164	183	299	324	356	388	400	3)
1.89	75	140	183	204	320	345	377	409	401	
1.89	80	150	132	152	268	294	325	358	389	
1.89	85	160	-	139	256	282	313	346	377	
1.91	95	180	-	-	231	257	289	321	353	

Small pulley effective diameter : $d_e = 75 \text{ mm}$
 Large pulley effective diameter : $D_e = 140 \text{ mm}$
 Center distance : $C = 401 \text{ mm}$
 Belt size = 3V-450

2
Design



Calculation example for Maxstar Wedge V-Belts #2 (With nonstandard pulley)

Design Flow

1

Set conditions required in design work.

- | | |
|--|---|
| a. Type of machine ... Generator | e. Speed ratio ... 1.65 (Deceleration) |
| b. Transmission power ... 37kW (Gasoline engine) | f. Interim center distance ... 1500mm |
| c. Running hours in a single day ... 8 hours / day | g. Special uses and environmental conditions ... None |
| d. Small pulley speed ... 1000rpm | |

Design Flow

2

Set the design power.

Service correction factor : $K_o = 1.3$ (Table 2-11)
 Idler correction factor : $K_i = 0$ (Table 2-12)
 Environment correction factor : $K_e = 0$ (Table 2-13)
 Service factor : $K_s = K_o + K_i + K_e$
 $= 1.3 + 0 + 0$
 $= 1.3$
 Design power : $P_d = P_t \times K_s$
 $= 37 \times 1.3$
 $= 48.1 \text{ kW}$

Design Flow

5

Determine the belt length.

1) Center distance = 1507 mm (5V-1500) is chosen from the SR = 1.67 in Drive Selection Table.

2) Precise center distance is calculated as follows.
 Effective diameter of the standard small pulley = 190 mm
 Effective diameter of the calculated small pulley = 192 mm

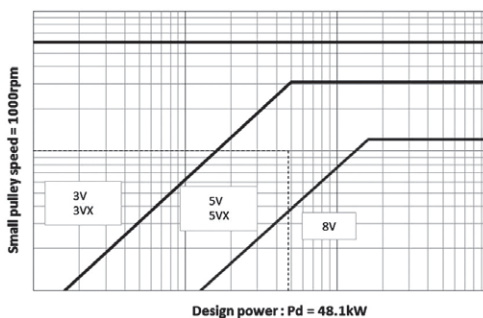
$$C = 1507 + 0.8 \times (190 - 192) \\ \div 1505 \text{ mm}$$

Design Flow

3

Select the belt type.

Select the belt type in the cross section selection chart.
 The lines of P_d (48.1kW) and small pulley speed (1000rpm) intersect in 5V section.



Design Flow

4

Select the pulley size.

- 1) Choose the speed ratio closest to the desired speed ratio.
- 2) Choose the large pulley from the standard pulley.

Large pulley effective diameter : $D_e = 315 \text{ mm}$
 is chosen from the SR = 1.67 in Drive Selection Table.

- 3) Calculate the small pulley effective diameter.

$$d_e = \frac{315 - 2.6}{1.65} + 2.6 \\ \div 192 \text{ mm}$$

Center distance is also calculated as follows.

$$b = 2L_e - \pi (D_e + d_e) \\ = 2 \times 3810 - 3.14 \times (315 + 192) \\ \div 6028 \\ C = \frac{b + \sqrt{b^2 - 8(D_e - d_e)^2}}{8} \\ = \frac{6028 + \sqrt{6028^2 - 8(315 - 192)^2}}{8} \\ \div 1505.7 \text{ mm}$$

Design Flow



Determine the required number of belts.

$P_s = 9.42 \text{ kW}$ (Refer to Power Rating Table)

$P_a = 0.85 \text{ kW}$ (Refer to Power Rating Table)

$$\frac{D_e - d_e}{C} = \frac{315 - 192}{1505} = 0.082$$

$K_\theta = 0.99$ (Refer to Table 2-16).

$K_\ell = 1.03$ (Refer to Table 2-17).

$K_c = K_\theta \times K_\ell = 0.99 \times 1.03 = 1.02$

$P_c = (P_s + P_a) \times K_c = (9.42 + 0.85) \times 1.02 = 10.48 \text{ kW}$

$$n_b = \frac{P_d}{P_c} = \frac{48.1}{10.48} = 4.6 \rightarrow 5 \text{ pcs.}$$

Design Flow



Installation and take-up allowance.

Installation and take-up allowance are obtained from Table 2-18.

Installation allowance = 25 mm

→ Minimum center distance = $1505 - 25 = 1480 \text{ mm}$

Take-up allowance = 60 mm

→ Maximum center distance = $1505 + 60 = 1565 \text{ mm}$

●Summary

Belt : 5V-1500, 5 pcs.

Driver pulley : 192-5V-5 (Effective diameter = 192mm, 5 grooves)

Driven pulley : 315-5V-5 (Effective diameter = 315mm, 5 grooves)

Center distance : 1505mm (+60mm / -25mm)



Formulas for V-Belt drives design

Table 2-19

Item	Formula	Term
Design power	$P_d = P_t \times K_s$	P_d : Design power (kW) P_t : Transmission power (kW) K_s : Service factor
Service factor	$K_s = K_o + K_i + K_e$	K_s : Service factor K_o : Service correction factor K_i : Idler correction factor K_e : Environment correction factor
Power rating	$P_r = P_s + P_a$	P_r : Power rating (kW) P_s : Basic power rating (kW) P_a : Additional power rating for speed ratio (kW)
Correction power rating	$P_c = P_r \times K_l \times K_\theta$	P_c : Correction power rating (kW) P_r : Power rating (kW) K_l : Belt length correction factor K_θ : Arc of contact correction factor
Speed ratio	$SR = \frac{n_d}{n_D} = \frac{D_p}{d_p}$	SR : Speed ratio n_d : Small pulley speed (rpm) n_D : Large pulley speed (rpm) D_p : Large pulley pitch diameter (mm) d_p : Small pulley pitch diameter (mm)
Interim effective length	$L_e' = 2C' + 1.57(D_e + d_e)$	L_e' : Interim effective length (mm) C' : Interim center distance (mm) D_e : Large pulley effective diameter (mm) d_e : Small pulley effective diameter (mm)
Effective length	$L_e = 2C + \frac{\pi(D_e + d_e)}{2} + \frac{(D_e - d_e)^2}{4C}$	L_e : Effective length (mm) C : Center distance (mm) D_e : Large pulley effective diameter (mm) d_e : Small pulley effective diameter (mm) π : 3.1416
Center distance	$C = \frac{b + \sqrt{b^2 - 8(D_e - d_e)^2}}{8}$ $b = 2L_e - \pi(D_e + d_e)$	C : Center distance (mm) D_e : Large pulley effective diameter (mm) d_e : Small pulley effective diameter (mm) L_e : Effective length (mm) π : 3.1416
Arc of contact	$\theta = 180^\circ - \frac{57.3(D_e - d_e)}{C}$	θ : Arc of contact for small pulley (°) D_e : Large pulley effective diameter (mm) d_e : Small pulley effective diameter (mm) C : Center distance (mm)
Number of belts	$n_b = \frac{P_d}{P_c}$	n_b : Number of belts P_d : Design power (kW) P_c : Correction power rating (kW)

Table 2-19

Item	Formula	Term
Belt speed	$V = \frac{\pi \times dp \times nd}{60 \times 1000} = \frac{dp \times nd}{19100}$	V : Belt speed (m/sec.) dp : Small pulley pitch diameter (mm) nd : Small pulley speed (rpm)
Transmission power	$Pt = \frac{Te \times V}{1000}$	Pt : Transmission power (kW) Te : Effective tension (N) V : Belt speed (m/sec.)
Transmission power	$Pt = \frac{Tq \times n}{9.55 \times 10^3}$	Pt : Transmission power (kW) Tq : Torque (N·m) n : Pulley speed (rpm)
Effective tension	$Te = \frac{2Tq}{dp} \times 1000$	Te : Effective tension (N) Tq : Torque (N·m) dp : Small pulley pitch diameter (mm)
Effective tension	$Te = \frac{1000 \times Pt}{V}$	Te : Effective tension (N) Pt : Transmission power (kW) V : Belt speed (m/sec.)
Torque	$Tq = Te \times \frac{dp}{2} \times \frac{1}{1000}$	Tq : Torque (N·m) Te : Effective tension (N) dp : Small pulley pitch diameter (mm)
Tight side tension	$Tt = \frac{1000 \times Pd}{nb \times V} \times \frac{2.5}{2 \times K\theta} + W \times V^2$	Tt : Tight side tension (N) Pd : Design power (kW) nb : Number of belts V : Belt speed (m/sec.) Kθ : Arc of contact correction factor W : Belt weight per unit (kg/m)
Slack side tension	$Ts = \frac{1000 \times Pd}{nb \times V} \times \frac{2.5 - 2 \times K\theta}{2 \times K\theta} + W \times V^2$	Ts : Slack side tension (N) Pd : Design power (kW) nb : Number of belts V : Belt speed (m/sec.) Kθ : Arc of contact correction factor W : Belt weight per unit (kg/m)
Tension ratio	$TR = \frac{Tt}{Ts}$	TR : Tension ratio Tt : Tight side tension (N) Ts : Slack side tension (N)
Minimum static tension	$To = 0.9 \left\{ 500 \times \frac{(2.5 - K\theta)Pd}{K\theta \times nb \times V} + W \times V^2 \right\}$	To : Minimum static tension (N) Kθ : Arc of contact correction factor Pd : Design power (kW) nb : Number of belts V : Belt speed (m/sec.) W : Belt weight per unit (kg/m)
Static shaft load	$Fs = 1.5 \left(2nb \times To \times \sin \frac{\theta}{2} \right)$	Fs : Static shaft load (N) nb : Number of belts To : Minimum static tension (N) θ : Arc of contact for small pulley (°)
Span length	$Ls = \sqrt{C^2 - \frac{(De - de)^2}{4}}$	Ls : Span length (mm) C : Center distance (mm) De : Large pulley effective diameter (mm) de : Small pulley effective diameter (mm)



● 3V·3VX (SR = 2.53 ~ 9.56)

Table 2-41-3 Drive Selection Table

Speed ratio	Effective diameter (mm)		Center distance (mm)													
	Small pulley	Large pulley	3V 250	3V 265	3V 280	3V 300	3V 315	3V 335	3V 355	3V 375	3V 400	3V 425	3V 450	3V 475	3V 500	3V 530
2.53	125	315	-	-	-	-	-	-	-	-	-	-	-	239	273	313
2.53	250	630	-	-	-	-	-	-	-	-	-	-	-	-	-	-
2.56	71	180	-	-	148	175	195	222	248	274	306	338	370	403	434	473
2.65	95	250	-	-	-	-	-	-	-	190	224	257	290	323	356	394
2.68	150	400	-	-	-	-	-	-	-	-	-	-	-	-	-	-
2.69	75	200	-	-	-	152	173	200	226	253	285	318	350	382	414	453
2.72	67	180	-	-	151	178	198	224	251	277	309	341	373	406	437	476
2.79	180	500	-	-	-	-	-	-	-	-	-	-	-	-	-	-
2.80	90	250	-	-	-	-	-	-	-	193	227	261	294	327	359	398
2.83	112	315	-	-	-	-	-	-	-	-	-	-	-	247	281	322
2.85	71	200	-	-	-	155	175	202	229	256	288	321	353	385	417	456
2.87	140	400	-	-	-	-	-	-	-	-	-	-	-	-	-	-
2.97	85	250	-	-	-	-	-	-	-	196	230	264	297	330	363	401
3.02	67	200	-	-	-	157	178	205	232	258	291	323	356	388	420	458
3.14	160	500	-	-	-	-	-	-	-	-	-	-	-	-	-	-
3.16	80	250	-	-	-	-	-	-	-	199	233	267	300	333	366	405
3.16	200	630	-	-	-	-	-	-	-	-	-	-	-	-	-	-
3.18	100	315	-	-	-	-	-	-	-	-	-	-	219	255	289	330
3.22	125	400	-	-	-	-	-	-	-	-	-	-	-	-	-	-
3.35	95	315	-	-	-	-	-	-	-	-	-	-	222	258	292	333
3.35	150	500	-	-	-	-	-	-	-	-	-	-	-	-	-	-
3.37	75	250	-	-	-	-	-	-	174	202	237	271	304	337	369	408
3.52	180	630	-	-	-	-	-	-	-	-	-	-	-	-	-	-
3.53	90	315	-	-	-	-	-	-	-	-	-	-	225	261	295	336
3.56	71	250	-	-	-	-	-	-	176	205	239	273	306	340	372	411
3.59	140	500	-	-	-	-	-	-	-	-	-	-	-	-	-	-
3.60	112	400	-	-	-	-	-	-	-	-	-	-	-	-	-	-
3.74	85	315	-	-	-	-	-	-	-	-	-	-	228	264	299	339
3.78	67	250	-	-	-	-	-	-	179	207	242	276	309	342	375	414
3.96	160	630	-	-	-	-	-	-	-	-	-	-	-	-	-	-
3.98	80	315	-	-	-	-	-	-	-	-	-	-	231	267	302	343
4.03	125	500	-	-	-	-	-	-	-	-	-	-	-	-	-	-
4.04	100	400	-	-	-	-	-	-	-	-	-	-	-	-	-	-
4.23	150	630	-	-	-	-	-	-	-	-	-	-	-	-	-	-
4.25	75	315	-	-	-	-	-	-	-	-	-	-	234	271	305	346
4.25	95	400	-	-	-	-	-	-	-	-	-	-	-	-	-	-
4.49	90	400	-	-	-	-	-	-	-	-	-	-	-	-	-	-
4.50	71	315	-	-	-	-	-	-	-	-	-	-	237	273	308	348
4.50	112	500	-	-	-	-	-	-	-	-	-	-	-	-	-	-
4.53	140	630	-	-	-	-	-	-	-	-	-	-	-	-	-	-
4.76	85	400	-	-	-	-	-	-	-	-	-	-	-	-	-	-
4.77	67	315	-	-	-	-	-	-	-	-	-	202	239	276	310	351
5.05	100	500	-	-	-	-	-	-	-	-	-	-	-	-	-	-
5.06	80	400	-	-	-	-	-	-	-	-	-	-	-	-	-	-
5.08	125	630	-	-	-	-	-	-	-	-	-	-	-	-	-	-
5.32	95	500	-	-	-	-	-	-	-	-	-	-	-	-	-	-
5.40	75	400	-	-	-	-	-	-	-	-	-	-	-	-	-	-
5.62	90	500	-	-	-	-	-	-	-	-	-	-	-	-	-	-
5.68	112	630	-	-	-	-	-	-	-	-	-	-	-	-	-	-
5.71	71	400	-	-	-	-	-	-	-	-	-	-	-	-	-	249
5.95	85	500	-	-	-	-	-	-	-	-	-	-	-	-	-	-
6.06	67	400	-	-	-	-	-	-	-	-	-	-	-	-	-	251
6.33	80	500	-	-	-	-	-	-	-	-	-	-	-	-	-	-
6.36	100	630	-	-	-	-	-	-	-	-	-	-	-	-	-	-
6.70	95	630	-	-	-	-	-	-	-	-	-	-	-	-	-	-
7.76	75	500	-	-	-	-	-	-	-	-	-	-	-	-	-	-
7.08	90	630	-	-	-	-	-	-	-	-	-	-	-	-	-	-
7.15	71	500	-	-	-	-	-	-	-	-	-	-	-	-	-	-
7.50	85	630	-	-	-	-	-	-	-	-	-	-	-	-	-	-
7.58	67	500	-	-	-	-	-	-	-	-	-	-	-	-	-	-
7.98	80	630	-	-	-	-	-	-	-	-	-	-	-	-	-	-
8.52	75	630	-	-	-	-	-	-	-	-	-	-	-	-	-	-
9.01	71	630	-	-	-	-	-	-	-	-	-	-	-	-	-	-
9.56	67	630	-	-	-	-	-	-	-	-	-	-	-	-	-	-

● 5V·5VX (SR =3.81 ~ 8.46)

Table 2-42-4 Drive Selection Table

Speed ratio	Effective diameter (mm)		Center distance (mm)															
	Small pulley	Large pulley	5V	5V	5V	5V	5V	5V	5V	5V	5V	5V	5V	5V	5V	5V	5V	5V
			500	530	560	600	630	670	710	750	800	850	900	950	1000	1060	1120	1180
1.00	300	300	-	-	-	-	-	-	-	-	-	-	-	-	-	-	549	636
1.00	315	315	-	-	-	-	-	-	-	-	-	-	463	534	604	685	766	845
1.00	335	335	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-
1.00	355	355	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-
1.00	375	375	-	-	-	-	-	-	-	-	-	-	-	-	-	-	556	643
1.00	400	400	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-
1.00	425	425	-	-	-	-	-	-	-	-	-	-	469	541	610	692	773	852
1.00	450	450	-	-	-	-	-	-	-	-	-	-	-	-	-	-	562	649
1.00	475	475	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-
1.00	500	500	-	-	-	-	-	-	-	-	-	-	-	-	-	-	568	656
1.00	560	560	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-
1.00	630	630	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-
1.00	710	710	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-
1.00	800	800	-	-	-	-	-	-	-	-	-	-	-	-	-	-	574	662
1.05	300	315	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-
1.05	475	500	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-
1.06	315	335	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-
1.06	335	355	-	-	-	-	-	-	-	-	-	-	-	-	-	-	580	668
1.06	355	375	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-
1.06	400	425	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-
1.06	425	450	-	-	-	-	-	-	-	-	-	-	-	-	-	493	586	674
1.06	450	475	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-
1.07	375	400	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-
1.11	450	500	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-
1.12	300	335	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-
1.12	335	375	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-
1.12	425	475	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-
1.12	500	560	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-
1.13	315	355	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-
1.13	355	400	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-
1.13	400	450	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-
1.13	560	630	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-
1.44	315	450	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-

5V·5VX**3.81 ~ 8.46**

Color coding of Power rating correction factor : Kc

0.7	0.8	0.9	1.0	1.1	1.2
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Center distance (mm)																			Speed ratio
5V 1250	5V 1320	5V 1400	5V 1500	5V 1600	5V 1700	5V 1800	5V 1900	5V 2000	5V 2120	5V 2240	5V 2360	5V 2500	5V 2650	5V 2800	5V 3000	5V 3150	5V 3350	5V 3550	
734	830	937	1070	1201	1332	1462	1591	1720	1875	2029	2182	2362	2554	2745	3001	3192	3447	3702	3.81
938	1029	1133	1263	1392	1520	1649	1777	1905	2059	2212	2365	2544	2735	2926	3181	3372	3626	3881	3.99
-	-	-	-	-	-	941	1083	1221	1384	1545	1704	1888	2084	2279	2538	2731	2989	3246	3.99
-	-	695	839	978	1114	1248	1380	1512	1669	1825	1980	2161	2354	2547	2803	2995	3251	3507	4.03
741	837	945	1078	1209	1340	1470	1599	1729	1883	2038	2191	2371	2563	2754	3010	3201	3456	3711	4.04
-	-	-	-	-	801	950	1092	1231	1394	1555	1714	1898	2094	2289	2548	2742	3000	3256	4.19
944	1036	1140	1270	1399	1528	1656	1784	1912	2066	2219	2372	2551	2742	2934	3188	3379	3634	3888	4.26
748	844	952	1085	1216	1347	1477	1607	1736	1890	2045	2198	2378	2570	2762	3017	3208	3464	3718	4.26
-	-	704	848	987	1123	1257	1390	1521	1678	1834	1990	2171	2364	2557	2813	3005	3261	3517	4.27
754	850	958	1091	1223	1354	1484	1614	1743	1897	2052	2206	2385	2577	2769	3024	3216	3471	3726	4.49
-	-	711	856	995	1131	1265	1398	1529	1687	1843	1998	2179	2372	2565	2822	3014	3270	3526	4.50
-	-	-	-	-	813	962	1105	1244	1407	1568	1727	1912	2108	2303	2562	2756	3014	3271	4.50
-	-	-	-	-	821	971	1114	1253	1417	1578	1737	1922	2118	2314	2573	2767	3025	3282	4.75
760	857	965	1098	1230	1361	1491	1621	1750	1905	2059	2213	2392	2584	2776	3032	3223	3478	3733	4.76
-	-	718	863	1003	1139	1273	1406	1538	1695	1851	2006	2188	2381	2574	2831	3023	3279	3535	4.76
-	-	-	-	-	830	980	1124	1263	1427	1588	1747	1932	2129	2324	2584	2777	3035	3292	5.04
-	-	725	871	1010	1147	1281	1414	1546	1703	1859	2015	2196	2390	2583	2839	3032	3288	3543	5.05
767	863	971	1105	1237	1368	1498	1628	1757	1912	2066	2220	2400	2592	2784	3039	3231	3486	3741	5.07
-	607	731	877	1017	1153	1288	1421	1553	1710	1866	2022	2203	2397	2590	2847	3039	3295	3551	5.32
-	-	-	-	-	839	989	1132	1272	1436	1597	1757	1942	2138	2334	2593	2787	3045	3302	5.34
773	870	978	1111	1243	1374	1505	1635	1764	1919	2073	2227	2407	2599	2791	3047	3238	3493	3748	5.41
-	613	737	883	1023	1160	1294	1427	1559	1717	1873	2029	2210	2404	2597	2854	3046	3302	3558	5.62
-	-	-	-	-	846	996	1140	1279	1444	1605	1765	1950	2147	2342	2602	2796	3054	3311	5.63
-	618	743	889	1029	1166	1301	1434	1566	1724	1880	2036	2217	2411	2604	2861	3053	3310	3565	5.96
-	-	-	-	-	853	1004	1147	1287	1451	1613	1773	1958	2155	2350	2610	2804	3062	3320	5.96
-	-	-	-	-	860	1011	1155	1295	1459	1621	1781	1966	2163	2359	2619	2813	3071	3328	6.32
-	624	749	895	1036	1173	1307	1441	1573	1730	1887	2043	2224	2418	2611	2868	3061	3317	3573	6.34
-	-	-	-	-	866	1017	1161	1301	1466	1628	1787	1973	2170	2366	2626	2820	3078	3335	6.66
-	630	755	902	1042	1179	1314	1447	1580	1737	1894	2050	2231	2425	2618	2875	3068	3324	3580	6.77
-	-	-	-	-	872	1023	1167	1307	1472	1634	1794	1980	2177	2373	2633	2827	3085	3343	7.03
-	-	-	-	-	878	1029	1173	1314	1479	1641	1801	1986	2183	2379	2639	2834	3092	3350	7.45
-	-	-	-	718	883	1035	1180	1320	1485	1647	1807	1993	2190	2386	2646	2841	3099	3357	7.93
-	-	-	-	723	889	1041	1186	1326	1492	1654	1814	2000	2197	2393	2653	2848	3106	3364	8.46



● 8V (SR = 4.04 ~ 5.41)

Table 2-43-3 Drive Selection Table

Speed ratio	Effective diameter (mm)		Center distance (mm)													
	Small pulley	Large pulley	8V	8V	8V	8V	8V	8V	8V	8V	8V	8V	8V	8V	8V	8V
			1000	1060	1120	1180	1250	1320	1400	1500	1600	1700	1800	1900	2000	2120
4.04	400	1600	-	-	-	-	-	-	-	-	-	-	-	-	-	-
4.22	300	1250	-	-	-	-	-	-	-	-	-	801	950	1092	1231	1394
4.31	375	1600	-	-	-	-	-	-	-	-	-	-	-	-	-	-
4.56	355	1600	-	-	-	-	-	-	-	-	-	-	-	-	-	-
4.83	335	1600	-	-	-	-	-	-	-	-	-	-	-	-	-	-
5.15	315	1600	-	-	-	-	-	-	-	-	-	-	-	-	-	977
5.41	300	1600	-	-	-	-	-	-	-	-	-	-	-	-	-	986



8V

4.04 ~ 5.41

Color coding of Power rating correction factor : Kc

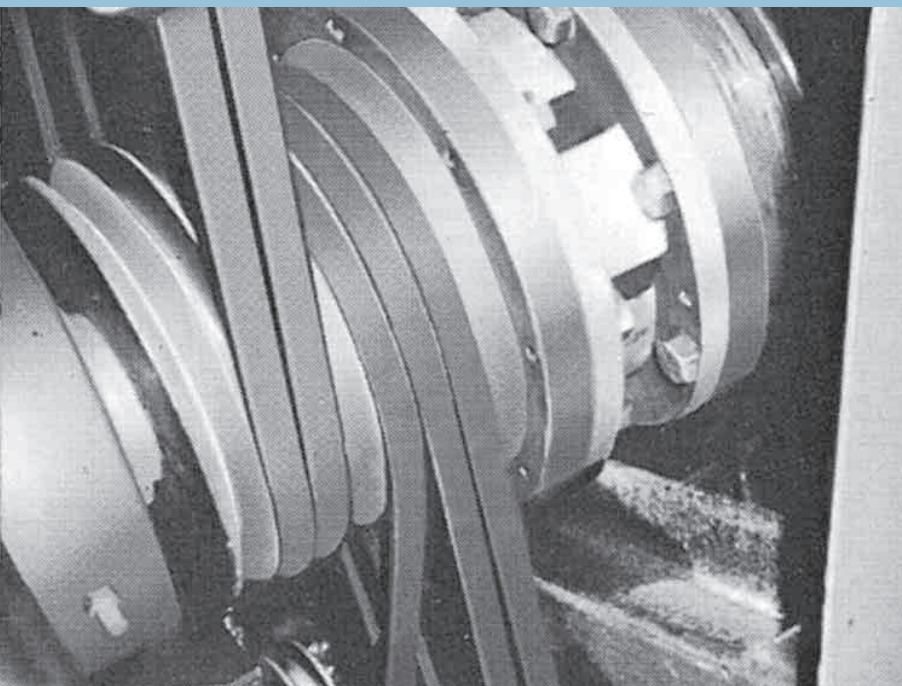
0.7	0.8	0.9	1.0	1.1	1.2
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Center distance (mm)																Speed ratio
8V 2240	8V 2360	8V 2500	8V 2650	8V 2800	8V 3000	8V 3150	8V 3350	8V 3550	8V 3750	8V 4000	8V 4250	8V 4500	8V 4750	8V 5000	8V 5600	
1112	1286	1483	1688	1890	2156	2353	2615	2875	3134	3457	3779	4100	4421	4741	5509	4.04
1555	1714	1898	2094	2289	2548	2742	3000	3256	3513	3833	4153	4472	4792	5111	5875	4.22
1127	1302	1499	1704	1906	2172	2370	2632	2892	3152	3475	3797	4118	4439	4759	5527	4.31
1140	1314	1511	1717	1920	2186	2384	2646	2906	3166	3489	3811	4133	4454	4774	5542	4.56
1152	1326	1524	1730	1933	2199	2397	2660	2920	3180	3503	3825	4147	4468	4788	5556	4.83
1164	1339	1537	1743	1946	2213	2411	2673	2934	3194	3517	3840	4161	4482	4803	5571	5.15
1173	1348	1546	1753	1956	2223	2421	2684	2944	3204	3528	3850	4172	4493	4814	5582	5.41



Reference





3. Reference

Length measurement

Tensioning

Use of idlers

Quarter-Turn Drives

V-Flat pulley drives

Multi V-Belts

Storage and Handling of V-Belts

Request for belt design

Global Factories & Sales Offices

Length measurement

Precise measuring methods are specified in DIN 2215 for Classical V-Belts, RMA IP-22 for Maxstar Wedge V-Belts, and DIN 7753 Part 1 for Narrow V-Belts. The V-Belt is laid over two equal size pulleys as following figure. These pulley grooves are designed to correspond with the belt section specified in Table 3-2 ~ 3-4. The measuring force is added to the measuring pulley in such a way. Belt length is calculated by the formula which is specified in Table 3-1.

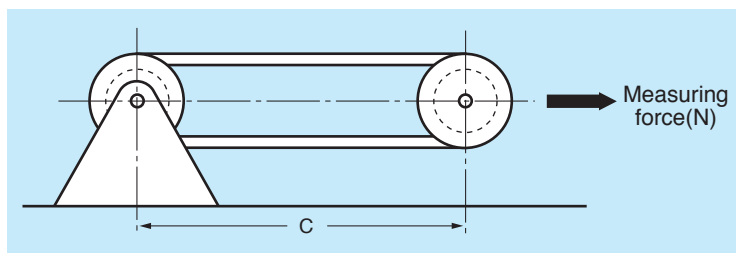


Fig. 3-1 Diagram of fixture for length measurement

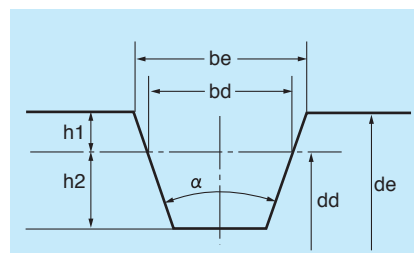


Fig. 3-2 Measuring pulley

Length calculation formula

Table 3-1

Belt section	Length designation	Length calculation formula	
Z, ZX	Inner length : L_i	$L_d = 2C + \pi dd$	$L_i = L_d - 22$
A, AX			$L_i = L_d - 30$
B, BX			$L_i = L_d - 43$
C, CX			$L_i = L_d - 52$
D			$L_i = L_d - 75$
E			$L_i = L_d - 82$
3V, 5V, 8V 3VX, 5VX	Effective length : L_e	$L_e = 2C + \pi de$	
SPZ, SPA, SPB, SPC SPZX, SPAX, SPBX, SPCX	Datum length : L_d	$L_d = 2C + \pi dd$	



Pulley dimensions and measuring force of Classical V-Belts for DIN**Table 3-2**

Belt section		Z ZX	A AX	B BX	C CX	D	E
Datum width : bd	mm	8.5	11.0	14.0	19.0	27.0	32.0
Datum diameter : dd	mm	57.30	95.50	127.32	222.82	318.31	572.96
Effective diameter : de	mm	62.30	102.10	135.72	234.22	334.51	596.96
Pulley datum circumference : πdd	mm	180	300	400	700	1000	1800
Angle : α	°	34	34	34	34	36	36
Distance down to datum line : h1	mm	2.5	3.3	4.2	5.7	8.1	12.0
Measuring force	N	110	200	300	750	1400	1800

**Pulley dimensions and measuring force of
Maxstar Wedge V-Belts for RMA/MPTA****Table 3-3**

Belt section		3V 3VX	5V 5VX	8V
Effective width : be	mm	8.9	15.2	25.4
Effective diameter : de	mm	95.5	191.0	318.3
Pulley effective circumference : πde	mm	300	600	1000
Angle : α	°	38	38	38
Measuring force	N	445	1000	2225

Pulley dimensions and measuring force of Narrow V-Belts for DIN**Table 3-4**

Belt section		SPZ SPZX	SPA SPAX	SPB SPBX	SPC SPCX
Datum width : bd	mm	8.5	11.0	14.0	19.0
Datum diameter : dd	mm	95.49	143.24	190.99	318.31
Effective diameter : de	mm	100	149	198	328
Pulley datum circumference : πdd	mm	300	450	600	1000
Angle : α	°	36	36	36	36
Distance down to datum line : h1	mm	2.255	2.880	3.505	4.845
Measuring force	N	360	560	900	1500



■ Tensioning design V-Belts

Tension of the belts on a V-Belt drive is usually not critical. A few simple rules about tensioning will satisfy most of your requirements. For your proper tensioning of V-Belts, just follow these four steps.

Step 1.

After placing V-Belts into the pulleys grooves, increase the distance between pulleys until V-Belts are snug.

Step 2.

Measure the length of the span for your drive. At the center of the span, apply Deflection load ($F\delta$) in Fig. 3-3 on page 3-7 with spring scale in a direction perpendicular to the span until the belt is deflected from the normal by amount equal to 1.6mm for every 100mm.

Step 3.

A few days are necessary for V-Belts to seat into pulley grooves. The belt tension for a V-Belt drive is the lowest at which the belts will not slip under the highest load condition. A bigger tension than force maximum will reduce the life of belts and bearings, and a less tension than force minimum, will cause slip.

Step 4.

During the normal operation, V-Belts will seat itself into pulley grooves, and will require periodic checks to maintain tension. The seating occurs more rapidly during the first 20-24 hours of operation. It is necessary to keep the belts and pulleys from any foreign materials which may cause slip. If V-Belts slip, tighten them. Recommendable belt Deflection load to get the proper tension is shown in table 3-6. But the ideal tension can be obtained as follows :

1. Calculate Span length

$$L_s = \sqrt{C^2 - \frac{(D_d - d_d)^2}{4}}$$

L_s : Span length (mm)

C : Center distance (mm)

D_d : Large pulley datum diameter (mm)

d_d : Small pulley datum diameter (mm)

2. Calculate Minimum static tension

$$T_o = \left\{ 500 \cdot \frac{(2.5 - K_\theta)}{K_\theta} \cdot \frac{P_d}{n_b \cdot V + WV^2} \right\} \times 0.9$$

T_o : Minimum static tension (N / pc.)

n_b : Number of belts

K_θ : Arc of contact correction factor

W : Belt weight per unit (kg / m) → see Table 3-5 in page 3-6

P_d : Design power (kW)

V : Belt speed (m/sec.)

$T_{\text{max}}(\text{initial})$: Maximum belt tension at initial fitting (N/a belt)

$T_{\text{max}}(\text{retension})$: Maximum belt tension at retensioning (N/a belt)

$T_{\text{max}}(\text{initial}) = 1.5 \cdot T_o$

$T_{\text{max}}(\text{retension}) = 1.3 \cdot T_o$

3. Calculate Deflection load

A) Multiple V-Belts drivers :

$$F\delta_{\text{min}} = \frac{T_o + Y}{16}$$

$$F\delta_{\text{max}}(\text{initial}) = \frac{1.5 \cdot T_o + Y}{16}$$

$F\delta$: Deflection load
(N / a belt)

$$F\delta_{\text{max}}(\text{retension}) = \frac{1.3 \cdot T_o + Y}{16}$$

Y : a constant → see Table 3-5 in page 3-6

L : Belt length (mm)

L_s : Span length (mm)

B) Single V-Belts drivers :

$$F\delta_{\text{min}} = \frac{T_o + Y(L_s/L)}{16}$$

$$F\delta_{\text{max}}(\text{initial}) = \frac{1.5 \cdot T_o + Y(L_s/L)}{16}$$

$$F\delta_{\text{max}}(\text{retension}) = \frac{1.3 \cdot T_o + Y(L_s/L)}{16}$$

4. Calculate maximum Shaft load at initial fitting

$$F_s = 2n_b \times T_o \times \sin \frac{\theta}{2} \times 1.5$$

F_s : Static shaft load (N)

θ : Arc of contact for small pulley



Belt weight per unit (W) & Constant (Y)

Table 3-5

Belt Section	W(kg/m)	Y(N/pc)
Z	0.05	9.8
A	0.12	14.7
B	0.20	19.6
C	0.30	29.4
D	0.65	58.8
E	1.00	108
SPZ	0.08	19.6
SPA	0.13	26.1
SPB	0.21	39.2
SPC	0.37	71.3
3V	0.08	19.6
5V	0.23	39.2
8V	0.60	98.1
ZX	0.06	9.8
AX	0.11	14.7
BX	0.18	19.6
CX	0.33	29.4
SPZX	0.08	19.6
SPAX	0.11	26.1
SPBX	0.21	39.2
SPCX	0.36	71.3
3VX	0.07	19.6
5VX	0.20	39.2



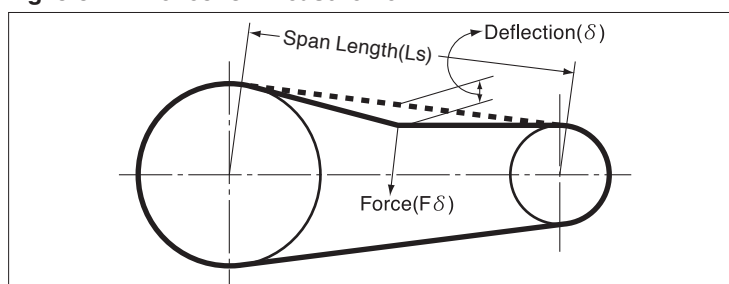
Tensioning of V-Belts

Step 1. Calculate Span length

Span length means the length between the belt contact points to the pulleys (L_s in Fig. 3-3).

If the diameter of the drive pulley and the driven pulley are the same, it is the same with Center Distance.

Fig. 3-3 Deflection measurement



Step 2. Calculate Deflection load by loading on the center of Span length.

Load on the center of Span length at right angle to the belt with the equipment like a spring balance, then check the load.

Deflection should be 1.6mm per 100mm of Span length. For example, if Span length is 500mm, Deflection should be 8mm.

Step 3. Adjust the belt tension so that the load calculated by step 2. is between maximum and minimum Deflection load in Table 3-6.

Deflection load & Belt tension

Table 3-6

Belt section	Small pulley diameter (mm)	Minimum tensioning conditions		Maximum tensioning conditions			
		Deflection load (N)	Belt tension (N)	Initial fitting		Retension	
				Deflection load (N)	Belt tension (N)	Deflection load (N)	Belt tension (N)
10/Z	~50	6	80	8	120	7	110
	51~70	7	110	11	160	9	140
	71~100	8	120	12	180	11	160
13/A	~80	12	170	17	250	15	220
	81~100	15	220	22	330	19	290
	101~132	18	270	27	410	23	350
17/B	~125	19	290	29	440	25	380
	126~160	25	380	36	560	32	490
	161~200	28	430	41	640	36	560
22/C	~200	36	540	52	800	46	700
	201~250	42	650	62	970	54	840
	251~355	50	770	74	1160	64	1000
32/D	~355	72	1090	106	1640	92	1420
	356~560	94	1450	139	2170	121	1880
	561~800	109	1690	162	2530	141	2190
40/E	~560	107	1600	157	2400	137	2080
	561~800	134	2030	197	3050	172	2640
	801~950	148	2260	219	3390	191	2940
SPZ/3V	~70	12	180	17	260	16	230
	71~90	15	220	22	340	19	290
	91~125	19	280	27	420	24	360
SPA	~100	20	290	29	430	25	370
	101~140	27	400	39	600	34	520
	141~200	32	480	47	720	40	620
SPB/5V	~160	36	530	52	800	46	690
	161~224	46	690	67	1030	58	890
	225~355	53	810	79	1220	69	1060
SPC	~250	66	990	98	1490	85	1290
	251~355	84	1280	124	1910	108	1660
	356~560	99	1510	146	2270	128	1970
8V	~355	104	1570	154	2360	134	2040
	356~560	136	2080	201	3120	175	2700
	561~800	154	2360	227	3540	198	3070
10/ZX	~50	7	110	11	170	10	150
	51~70	9	130	12	190	11	170
	71~100	9	140	14	210	12	180
13/AX	~80	17	250	24	370	21	320
	81~100	18	280	27	420	23	360
	101~132	20	310	30	460	26	400
17/BX	~125	27	410	40	620	35	540
	126~160	29	440	42	660	37	570
	161~200	30	460	44	690	39	600
22/CX	~200	46	710	69	1070	60	930
	201~250	48	740	71	1110	62	960
	251~355	50	770	74	1160	64	1000
SPZX/3VX	~70	17	260	26	390	22	340
	71~90	20	300	29	450	26	390
	91~125	22	340	33	510	29	440
SPAX	~100	27	400	39	600	34	520
	101~140	33	500	49	750	42	650
	141~200	37	570	55	860	48	740
SPBX/5VX	~160	46	690	67	1040	59	900
	161~224	55	840	81	1250	71	1090
	225~355	63	970	93	1450	81	1260
SPCX	~250	79	1200	117	1800	102	1560
	251~355	91	1380	134	2070	116	1790
	356~560	101	1550	150	2330	131	2020



■ Use of idlers

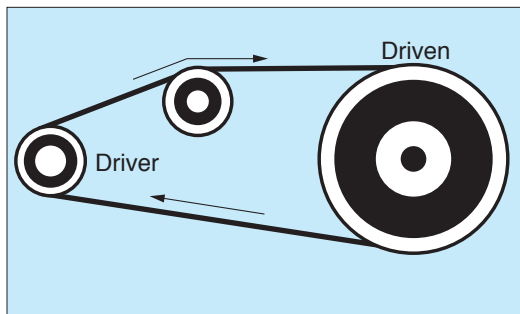
Idlers shorten the belt life. Use idlers only in the following cases.

- When the center distance cannot be adjusted.
- When the V-Belt is used as a clutch.
- When the belt span is too long and the belt vibrates.
- When a longer arc of contact with pulleys is required.
- When the belt tension is to be maintained during operation.
- When the belt is required to avoid obstructions.

Use of inside idlers

Inside idler

Fig. 3-4



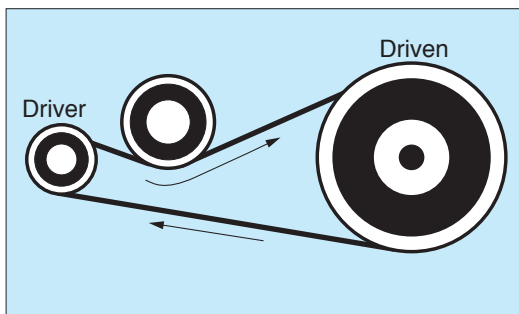
Place a grooved idler on the slack side of the belt. It is preferable to use an idler on the inside of the layout, not the outside.

The inside idler should be placed near the large pulley, otherwise the arc of contact becomes smaller and belt might slip.

Use of backside idlers

Backside idler

Fig. 3-5



The backside idlers shorten the belt life significantly, and are not recommended.

Use a flat pulley as a backside idler and place it near the small pulley.

The diameter of idler pulleys shall be larger than the diameter specified in Table 3-7.

Use the idler bigger than twice the size of the small pulley for Maxstar Wedge.

Minimum datum diameter of idler pulley

Table 3-7

Section	Inside idler(mm)	Backside idler(mm)
A	75	110
B	125	190
C	225	340
D	325	490
E	525	790

■ Quarter-Turn Drives

Quarter-Turn Drives are drives where the driver and driven shafts are at right angles to each other. To design Quarter-Turn Drives, follow the steps given in "Calculation of V-Belt drives design" section for designing an ordinary drive, keeping in mind the following special points:

1. Speed ratio should be 2.50 or less.
2. A standard V-Belt length should be chosen which will give a minimum Center distance of:

$$\text{Minimum } C = 5.5(D+W)(\text{mm})$$

$$D = \text{Large pulley outside diameter (mm)}$$

$$W = \text{Width of Deep Grooved Pulley, from Table 3-9}$$
3. Aligning the drive
 Looking down on the drive, a line from the center of the vertical shaft should pass through the center of the face of the pulley on the horizontal shaft. The horizontal shaft should be at right angles to this line. See "Top View" in Fig. 3-6.
 Looking at the side of the drive, the center of the horizontal shaft should be raised a distance "Y", from Table 3-8 above the level line through the center of the face of the pulley on the vertical shaft. See "Side View" in Fig. 3-6.
4. Direction of rotation
 The direction of rotation must be such that the Tight side of the drive will be on the bottom. See "Side View" in Fig. 3-6.
5. Power rating for Quarter-Turn Drives should be 90% from it for ordinary drives. And Arc of contact correction factor ($K\theta$) may be taken as 1.00 on Quarter-Turn Drives.
6. Deep grooved pulleys should always be used on Quarter-Turn Drives using individual V-Belts.

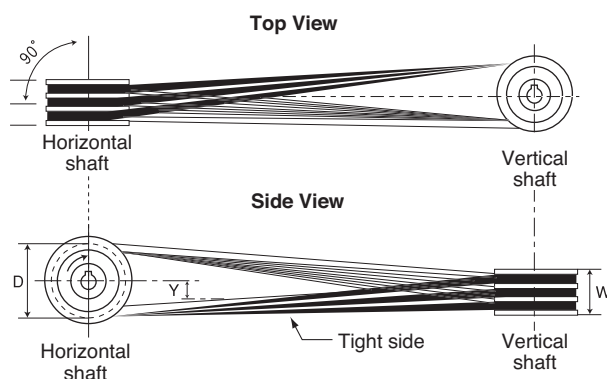


Fig. 3-6

Table 3-8 (unit: mm)

Center distance	Dimension Y
1500	65
2000	70
2500	75
3000	100
3500	130
4000	165
4500	195
5000	230
5500	270
6000	305

Width of Deep grooved pulley (W)

Table 3-9 (unit: mm)

No. of Grooves	V-Belt Section							
	A	B	C	D	E	3V	5V	8V
1	24	29	41	54	68	19	29	43
2	42	51	73	98	120.5	32	49	76
3	60	73	105	143	173	44	70	110
4	78	95	136	187	225.5	57	90	143
5	96	117	168	232	278	70	111	176
6	114	140	200	276	330.5	83	132	209
7	132	162	232	321	383	95	152	243
8	150	184	263	365	435.5	108	173	276
9	168	206	295	410	488	121	194	309
10	186	229	327	454	540.5	133	214	343

■ V-Flat pulley drives

What V-Flat drive is to use a V-Grooved pulley and the other flat pulley with regard to V-Belt drive.

This type of drive is used when it is desirable to change a flat belt drive into a V-Belt drive, because it is often most economical to retain the flat pulley.

Classical V-Belts for DIN (Z,A,B,C,D) are suitable for the drive.

The following prerequisites must be fulfilled to ensure the operating reliability of V-Belt drive:

- The small pulley must be a grooved pulley.
- Speed ratio must be $SR \geq 3$.
- Belt speed must be $V \leq 25$ m/sec.
- The datum diameter of flat pulley = Outside diameter + Value of table 3-11
- Relation between pulley diameter and Center distance must be fulfilled the following formula.

$$0.48 \leq \frac{Dd - dd}{C} \leq 1.17$$

Dd : Datum diameter of flat pulley (mm)

dd : Datum diameter of V pulley (mm)

C : Center distance (mm)

Table 3-10
Arc of contact correction factor for V-Belt drives

$\frac{Dd-dd}{C}$	Contact of small pulley $\theta (^\circ)$	Correction factor K_θ
0.00	180	0.75
0.10	174	0.76
0.20	169	0.78
0.30	163	0.79
0.40	157	0.80
0.50	151	0.81
0.60	145	0.83
0.70	139	0.84
0.80	133	0.85
0.90	127	0.85
1.00	120	0.82
1.10	113	0.80
1.20	106	0.77
1.30	99	0.73
1.40	91	0.70
1.50	83	0.65

Table 3-11
Difference between pulley datum diameter and outside diameter

Belt Section	Z	A	B	C	D	E
Difference between datum diameter and outside diameter (mm)	5.0	6.6	8.4	11.4	16.2	24.0

Multi V-Belts

▲ MITSUBOSHI MULTI (Banded) V-Belt is made up of two or more standard V-Belts connected together at the top as shown in the picture. No special pulleys are needed, as the individual belts have the same cross section and spacing as those which operate on standard pulleys. The top backing of Multi V-Belts does not come in contact with the top of the pulleys, so each multiple belt produces the same wedge effect as a single belt.

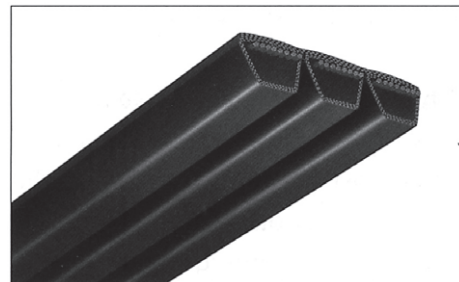


Fig. 3-7

Advantage of Multi (Banded) V-Belts

In most of the applications, V-Belts can meet the drive requirements. However, under certain operating conditions, belt whipping or vibration may become a critical problem, causing belts ultimately to come off the drive possible causes include the following:

- 1) Load vibration occurs periodically either on the driver side or at the drive unit, e.g. internal combustion engine, air compressor or piston pump.
- 2) There is excessively large load vibration or shock load, e.g. hoist or press.
- 3) Long span.
- 4) Vertical shaft length drives.

Belt vibration occurs laterally, as well as vertically. Under these conditions single matched sets of belts will be out of alignment in entering the pulley and will be damaged turned over or thrown off the drive. Multi V-Belts are recommended for use under these conditions as they can stand lateral stress, and belt vibration is virtually eliminated, resulting in longer belt life expectancy.

How to select Multi V-Belts

1) Size range

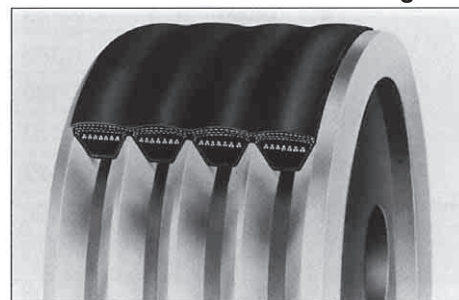
The available sizes are as follows.

Available range of Multi V-Belts

Table 3-12

Cross section	Range of available size	Center to center of pulley grooves(P)
B, BX	B60~B315	19mm
C, CX	C76~C720	25.4mm
D	D90~D600	36.5mm
3V, 3VX	3V600~3V1400	10.3mm
5V, 5VX	5V630~5V3550	17.5mm
8V	8V1000~8V5000	28.6mm
SPB	SPB2120~SPB10287	19.0mm
SPC	SPC2120~SPC10668	25.5mm

Fig. 3-8



2) Number of ribs

Multi V-Belts are available in 2,3,4 and 5 ribs.

They may be used in matched sets for drives requiring more than 5 belts, as shown in the following table.

Table 3-13

Number of belts	Recommended combination	Number of belts	Recommended combination	Number of belts	Recommended combination
2	2	9	5,4	16	4,4,4,4
3	3	10	5,5	17	4,4,5,4
4	4	11	4,3,4	18	4,4,5,5
5	5	12	4,4,4	19	5,4,5,5
6	3,3	13	4,5,4	20	5,5,5,5
7	3,4	14	5,4,5		
8	4,4	15	5,5,5		



Tensioning of Multi V-Belts

The usual tensioning method by Deflection load may not be usable for the accurate checking of tensioning of Multi V-Belts.

Initial tension can be alternatively checked by the elongation method as follows:

Step1

Find minimum static tension per a belt (T_o), using formula on page 2-20.

Then find the range of recommendation tension.

Minimum tension= T_o

Maximum tension $T_{max.} = 1.5 \times T_o$.

Step2

Find the amount of elongation for belt to obtain the above tensions.

·Measurement Outer length of the belt at no tension. This can be done with the belt either on or off the drive.

·Find the belt length multipliers from below table for the above tension.

·Multiply the measured Outer length of the belt by each belt length multiplier to obtain elongated outside circumference corresponding to each calculated tension.

Step3

Tense the drive until the measured outside circumference falls within the range of elongated minimum and maximum length values determined above.

Belt length multiplier for Classical V-Belts for DIN

Table 3-14

Calculated Tension T_o (N)	Cross section			Calculated Tension T_o (N)	Cross section		
	B	C	D		B	C	D
196	1.0026			883		1.009	1.0045
294	1.0039	1.003		981		1.010	1.0050
392	1.0053	1.004		1180			1.0060
490	1.0067	1.005		1370			1.0070
588	1.0080	1.006	1.0030	1570			1.0080
686	1.0093	1.007	1.0035	1770			1.0090
785	1.0107	1.008	1.0040	1960			1.0100

Belt length multiplier for Maxstar Wedge V-Belts for RMA / MPTA

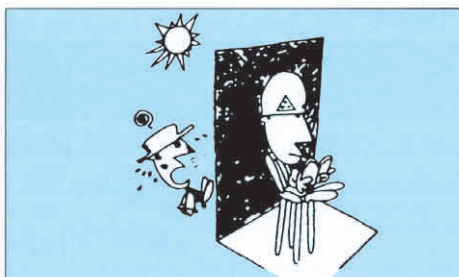
Table 3-15

Calculated Tension T_o (N)	Cross section			Calculated Tension T_o (N)	Cross section		
	3V	5V	8V		3V	5V	8V
177	1.00228			981		1.00648	1.00220
196	1.00265			1180		1.00819	1.00275
216	1.00303			1370		1.01003	1.00334
245	1.00361			1570		1.01201	1.00397
265	1.00400			1770		1.01412	1.00463
294	1.00459			1960		1.01637	1.00532
314	1.00500			2210		1.01937	1.00625
343	1.00561			2450			1.00723
363	1.00603			2700			1.00826
392	1.00667			2940			1.00936
412	1.00710			3190			1.01051
441	1.00775	1.00228		3430			1.01172
461	1.00819	1.00261		3680			1.01299
490	1.00887	1.00281		3920			1.01431
588	1.01120	1.00347		4170			1.01569
686	1.01365	1.00417		4410			1.01713
785	1.01624	1.00491		4900			1.01863
883	1.01896	1.00568					



■ Storage and Handling of V-Belts

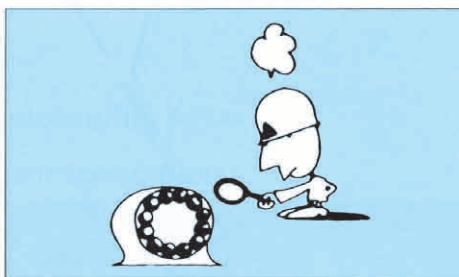
Storage of V-Belts



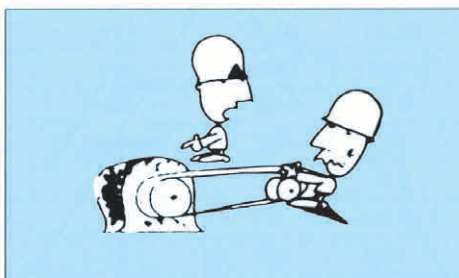
Do not expose belts direct to sunlight during storage. Store belts in shelves or hang belts on racks. Avoid placing belts on floor or ground. Avoid piling belts up. Avoid storing belts in heavily bent condition.

Keep belts away from oil and grease.

Installation of V-Belts

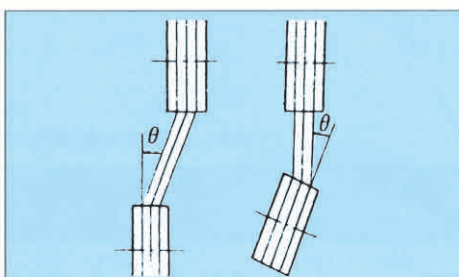


Check bearing for oil.



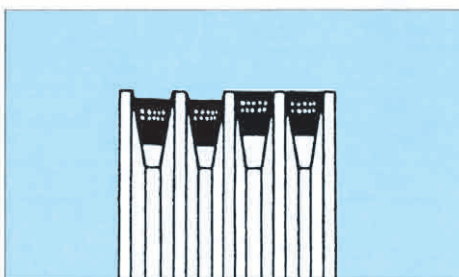
Slack off on take -up until belts can be placed in grooves without forcing.

Never pry the belts into the pulley grooves to prevent cord break.



Check pulley alignment. Misalignment of pulley will shorten the belt life.

Keep deflection angle less than $1/3^\circ$.



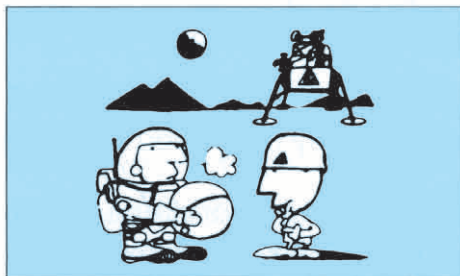
Make sure all pulley grooves are equal in dimensions.

Uneven pulley groove wear produces the same bad effect as mismatched belts.



Tension drive properly. See page 3-5~3-7 for belt tensioning method. Give belts a few day running time to become seated in pulley grooves, then readjust take-up if necessary

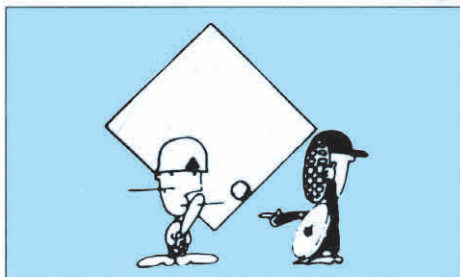
Maintenance



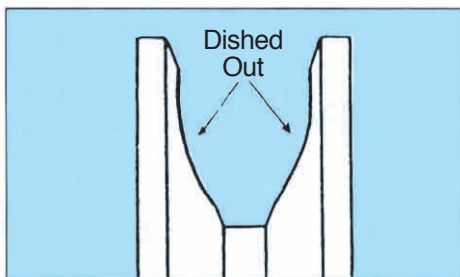
Make sure V-Belt drives are covered for safety before turning the machine on. An air tight cover shortens the belt life because it prevents heat radiation. Maintain proper ventilation.



Never use a belt dressing. Be careful not to expose belts to water, oil or chemicals.



Always use a set of new belts from one manufacture to replace a set of belts. Mixing new and used belts in a set will shorten the belt life due to unequal stretch of belts.



Check pulley for groove wear. If more than 1/32 inch (0.794mm) of "Dished Out" can be seen, short belt life may be expected.



Request for belt design

Date: _____

Customer's Name							
OEM/Replacement							
Purpose							
Model Name							
Drawing	available	not available	availability: yes / no (Date:)				
Specification of Belt							
Belt Size							
Number of Belts	pcs.						
Annual Quantity	pcs./year						
Operating Conditions	Type of Prime Mover	Power		kW		N•m	maximum torque
	Speed	Drive		rpm	Driven		rpm
	Pulley datum diameter	Drive		mm/teeth	Driven		mm/teeth
	Center distance		±		mm	Speed ratio	
	Operational Hours per day		hrs./day		Idler Pulley:		
	Other Special Conditions						
	Ambient Conditions						

Other Information

Information for designing	
Information for price	



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